



**CITY OF YUMA
FLEET SERVICES MAINTENANCE SHOP
265 W 13TH ST
YUMA, AZ 85364**

COST REVIEW STUDY

JULY 25, 2017

WSM
ARCHITECTS
4330 NORTH CAMPBELL AVE
SUITE NO 265
TUCSON, ARIZONA 85718
520.408.1044 TEL
520.408.1170 FAX
WSMARCH.COM WEB

thompson
design
architects
pc

Architect Project No. 15-054



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FLEET SERVICES MAINTENANCE SHOP



TABLE OF CONTENTS

ARCHITECTS

Introduction and Cost Summary

Option: Pre-Engineered Metal Building

Option: Masonry Exterior

Mechanical Systems

Shop Doors

Appendix

Structural Summary Pre-Engineed Metal Building Design

Review of fourfold door and Yuma Maintenance information

HVAC Mechanical Analysis Back-up

Alternate Design Drawings



INTRODUCTION

WSM Architects and Thompson Design Architects were asked to consider several options/ alternatives to the current design of the Fleet Maintenance Facility.

Those were:

- Review the costs and pros & cons associated with changing the current design of a conventional tilt-up concrete wall and steel joists and framing with a pre-engineered metal building structure.
- Review the costs associated with providing concrete block masonry walls instead of the tilt-up concrete walls.
- Review the costs associated and life cycle costs associated with an all evaporative cooling conditioning system for the shop space verses a hybrid evaporative / AC system.
- Review alternatives for the two main doors into the shop space in lieu of the four fold door specified.
- Below is a Summary of the cost differences of the options. Note there are performance differences between the options that are outlined in the balance of the report.
-

Pre-Engineered Metal Building Option vs Conventional Concrete Tilt & Steel Frame

Total Difference Construction Costs(savings to go to metal building) \$ 482,100

Annual difference in 40 year Cost of Ownership for metal exterior building \$ 3,321

Concrete Block Exterior walls as an alternate to the tilt up concrete.

Total Construction Cost Difference- Increase for Masonry (\$ 15,360.29)

Evaporative Cooling System for Shop space verses Hybrid Evap / AC.

Total difference Construction Costs (savings) for Alternate Evap Cooler system \$ 231,512

Annual Operating and Maintenance Cost difference for Evap Cooler system

\$ 4,186 or approx \$350/month savings \$ 11.50 per day

Total 20 year Life Cycle Annual cost of Ownership difference for Evap Cooler

\$ 14,042 or approx \$1170/month saving \$ 39 per day

Cost Comparison Shop Space Door Alternatives

16x16 Fourfold Door \$48,000 Specified Product

16x16 Sectional OH Door \$ 6,300

16x16 Coiling OH Door \$ 8,600

The annual maintenance savings for the four fold doors over sectional overhead doors for 2016 is \$6,370.

This equals a payback of 6.5 years for the fourfold door at the costs above.

The original bids received April 4, 2017 Escalation from this date to potential re-bid date is 2.9% annual escalation or 1.7% escalation for 7 months.

Approximate original bid amount \$7,800,000 x 1.7% = \$132,600, or \$ 7,932,600 adjusted bid number.

If the metal building and evap cooler options are selected

Approximate new construction estimate \$ 7,218,988



OPTION: PRE-ENGINEERED METAL BUILDING

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Pre-Engineered Metal Building Option (See attached drawings)

- The design was modified at a schematic level to provide a pre-engineered metal building structure. The framing of the building would be steel columns and beams with intermediate purlins supporting a metal panel skin and roof. Batt insulation is added to the interior face of the walls and hung from the metal roof to meet the energy code. The interior exposed face of the wall and roof would be a white vinyl plastic sheeting. This is the metal building standard.
- The existing floor plan was modified to accommodate the change to a pre-engineered metal building option. The main modifications to layout are:
 - Increased the width of the shop space due to the thickness of the new columns and to maintain the required clear floor space for safety and functional purposes.
 - The addition of a free standing structural frame to support the overhead bridge crane at the heavy truck repair bays.
- Due to the metal skin on the building the electrical engineer has recommended the installation of a lighting protection system for pre-engineered metal building. This would provide for the routing of lightning strikes to the ground through a designed path rather than through the building skin and frame.

Structural Pros and Cons

- Changing this facility from conventional construction to a pre-engineered metal building is a major change in systems and its design approach. Differences between the two systems:

Conventional Structure- Original Design - concrete tilt with steel joists and deck

- Easier to make additions to.
- Robust design for earthquake loads as well as future loads.
- Design can accommodate unique architectural features and column spacing.
- Cost of construction is often more than pre-engineered.
- Longer usable life of the building without major modifications to exterior envelope. Steel and concrete frame have a 100+ year service life.
- Roof system TPO membrane: Regular preventative maintenance would be required and replacement of roof membrane would be required at end of life. 20 year roof warranty provided in current design. The replacement of the roof membrane can be done without impacting day to day operations.
- Costs of concrete tilt panels were verified with a local concrete contractor with experience in this building type.

Pre-engineered Structure

- Thin steel roof decking that does not serve as a building diaphragm which cannot have anything hung from it.
- Building that is not easy to add future loads to or hang heavy items from the different parts of the building.
- Lateral system connections load paths are not direct.
- Lighter in weight so the earthquake has less mass to push around.
- Economy of design is uniform column spacing in the 25 feet on center range.
- Roof system: Structural sheet metal panels: Regular preventative maintenance would be required. 20 year weathertightness warranty can be provided. The replacement of the roof panels could impact day to day operations.
- The overhead crane will need its own structure since interior columns are not needed for PEMB and column spacing does not accommodate vehicle repair spaces.
- Cost of Construction for the building envelope is often less than that of convention systems.
- Long Term Issues for Pre-engineered building
- Thin metal roof and wall panels are easily damaged. Less durable than the conventional wall and roof structures. The conventional concrete walls have a 100+ year service life. The metal walls and roof 30 to 40 years.



OPTION: PRE-ENGINEERED METAL BUILDING

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- Slab on grade modifications and cutting is difficult due to thrust bars cast into the slab required to support the building.
- Difficult to add heavy loads to the structure like unforeseen hose reels or equipment. However, we will require higher collateral loads to the building to help minimize this issue. Note that this will add cost to the typical PEMB.
- Harder to add building additions to them.
- Weather tightness warranty impacted with additional loads to structure.
- Costs for the Pre-engineered building components were verified with a local metal building contractor.

Cost Comparison

	Original Design Concrete Tilt-up	Metal Building
Substructure	\$ 366,227	\$ 361,360
Superstructure	\$ 391,337	\$ 5,568
Exterior Closure	\$ 760,898	\$ 210,958
Roofing	\$ 396,785	\$ 11,386
Special Construction (Metal Building)	\$ 0	\$ 832,400
Lightning Protection system \$0.66/sf	\$ 0	\$ 28,036
Subtotal	\$1,915,247	\$1,449,708
Difference in Construction Cost		\$ 465,539
Contractor Markup (24.4%)		\$ 113,591
Subtotal		\$ 579,130
Design Fees to change to metal building design & re-bid		\$ 97,030
Total Delta Difference (savings)		\$ 482,100

Cost of Ownership - Life Cycle Costs

Metal building vs Conventional building (Concrete Tilt up- Metal Joists & Deck)

In comparing the life cycle costs of the two options there are a couple of assumptions for each system.

1. The foundations and substructure are designed to last the lifetime of the building.
2. The main structural frame, beams columns joists and decking are designed to last the lifetime of the building.
3. The doors and windows in the exterior of the building would be the same in either case so there would be no difference in life cycle costs for those elements.
4. Energy usage costs would be the same for either option as the insulation value would be the same for each.

The above assumptions being a given, the systems to be compared between the metal building and conventional building would be the exterior envelope. In the conventional building, this would be the concrete exterior walls and the roofing installed on top of the metal decking. In the metal building, this would be the metal siding and metal roofing.

**Estimated installed costs and cost per year***

<u>Original Design</u>	Quantity	cost/unit	total first cost	Ave life cycle	Annual cost / 40 years
Concrete walls	22,135 sf	\$16.25	\$ 407,452	80 years (lifetime)	\$ 5,093
TPO roof	39,923 sf	\$ 6.69	\$ 267,084	20 years	\$ 13,354
Subtotal Envelope Costs			\$ 674,536		\$ 18,447
<u>Metal Building Option</u>					
Metal roofing	39,923 sf	\$ 9.75	\$ 389,249	40 years	\$ 9,731
Metal siding	22,135 sf	\$ 9.75	\$ 215,816	40 years	\$ 5,395
Subtotal Envelope Costs			\$ 605,065		\$ 15,126
Difference Between options			\$ 69,471	Annual difference	\$ 3,321

*This assumes a 40 year timeframe and replacement costs in today's dollars

The metal roofing and siding would need to be replaced at the end of the 40 years while the concrete walls would not need replacement during the life of the building. The concrete and metal siding have a similar cost of ownership over 40 years but the concrete walls are more durable than the metal siding. Use beyond the 40 years would require replacement of the metal siding and roofing and would increase the ownership costs.

OPTION: MASONRY**Concrete Block Exterior walls as an alternate to the tilt up concrete.**

As the thickness and design of a masonry or concrete block wall is essentially the same as the concrete wall there are limited impacts to the layout and operation of the facility design. The pricing used for the tilt concrete came from a local Yuma concrete contractor as did the costs for the concrete block. Our estimate included some textured block similar to the texture provided in the concrete wall. There would be limited modifications to the footings and foundation. Per the estimated costs between the two, the concrete block is fractionally more expensive. The difference between the two is not significant and they are essentially the same price.

Cost Comparison

TU concrete Panels 25,074 SF at \$16.25	\$407,452.50
8" Gray Fluted CMU 9,100 SF at \$18.00	\$163,800.00
8" Gray Smooth CMU 16,000 SF at \$16.00	\$256,000.00
Total:	\$419,800.00
Delta:	(\$12,347.50)
Contractor Markup (24.4%)	(\$ 3,012.79)

Total Difference- Increase for Masonry (\$15,360.29)

**MECHANICAL SYSTEMS****ARCHITECTS****Evaporative Cooling System for Shop space verses Hybrid Evap / AC.** (See attached drawings)

The current design for the conditioning of the shop space is a hybrid unit that utilizes a combination of indirect and direct evaporative cooling (IEC/DEC) supplemented by mechanical cooling (DX). During those times of the year when humidity is low, the IEC/DEC will provide comfortable conditions very economically. During the monsoon season, the hybrid design uses the indirect evaporative cooling (IEC) and mechanical cooling (DX) as required to achieve the desired temperature and humidity levels. The integral control system monitors room temperature and humidity levels and will turn off the Direct Evaporative Cooling (DEC) when space humidity levels get too high to avoid adding moisture into the building, which will improve employee comfort. The unit includes integral gas heat when needed. The code required exhaust system is built into the unit, which also employs air-to-air energy recovery to reduce operating costs in both heating and cooling modes. The system is designed to maintain the shop in the 72oF (hot & dry conditions) to 80oF (monsoon conditions) range with a maximum humidity level of 55%. These conditions roughly produce a heat index of 80 degrees.

The alternative to this system that we were asked to look at was an all direct evaporative cooled (DEC) space. This system will provide reasonably effective cooling during the low humidity months but will be ineffective during periods of high humidity when indoor temperature and humidity will increase potentially causing discomfort. During these high humidity conditions, the shop temperature will most-likely be in the 80oF to 90oF range with a humidity level pushing 70%. These conditions roughly produce a heat index of 95 degrees.

The 2-speed evaporative coolers will be outfitted with natural gas heaters to maintain comfortable temperatures during the winter season. The code required exhaust system has been added as a separate system without energy recovery to ensure an equivalent comparison.

The cost comparison analysis and 20 year life cycle was done using the equipment costs for the two options and did not include ductwork and other items that would have a longer lifespan than the equipment. It would be anticipated that the internal mechanical components would need to be replaced at the 20 year mark and not the entire unit.

Please see the attached letter and back-up documentation for additional information on the calculations.

Cost Comparison:

Original Total Hybrid AHU System	\$ 589,620
Alternate Evaporative Cooler system	\$ 392,500
Difference in first cost	\$ 197,120
Contractor Markup (24.4%)	\$ 48,097
Subtotal	\$ 245,217
Mechanical system re-design costs	\$ 13,705
Total difference for Alternate Evap Cooler system	\$231,512
Annual Operating and Maintenance Cost difference	\$ 4,186 or approx \$350/month savings \$11.50 per day
Total 20 year life cycle annual cost of Ownership difference	\$ 14,042 or approx \$1170/month saving \$39 per day

The increased cost of operation and ownership should be weighed against the potential decrease in productivity due to the higher temperatures and humidity during certain times of the year.



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MECHANICAL SYSTEMS

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Date: July 21, 2017

Address: Mr. Paul Mickelberg
WSM Architects
4330 N. Campbell Ave, Suite 268
Tucson, AZ 85718

RE: 15241 Yuma Fleet Services – HVAC Alternative

Dear Mr. Mickelberg:

This letter is to address the difference in cost of ownership between the proposed design and the alternate design. The total cost of ownership includes the first cost, utility cost, and maintenance cost. The maintenance cost includes the evaporative pad replacement cost. Both manufacturers recommend pad replacement every eight years. Therefore, the total pad replacement cost was spread over eight years, and incorporated into the annual maintenance cost. However, in the “cost of ownership” table a line item called “Pad Replacement Cost” was provided so that the reviewer has a feel for the cost associated with pad replacement.

The proposed system provides conditioned air for the shop space. AHU-1 is a hybrid unit that provides evaporative cooling (direct & indirect) through those times of the year when humidity is low and evaporative cooling is most effective. During the monsoon season and sometimes during the fall and winter humidity levels rise to a point where evaporative cooling is no longer effective. The hybrid design then uses the indirect evaporative cooling section to precool the air and then uses an AC section (DX coil) to cool the air to the desired temperature. AHU-1 also employs energy recovery to extract usable energy from the exhaust air stream prior to discharging it to atmosphere to optimize the efficiency of the unit. The energy recovery unit is the reason why there is a substantial difference between the proposed and alternate systems in natural gas cost. The alternate system doesn’t have an energy recovery unit and therefore, it uses more natural gas during the heating cycle. See the “cost of ownership” table below for the total cost of the proposed design system. Note that the proposed construction cost of \$1,162,798, which includes all the HVAC systems for the entire design, minus General Contractor markup. This includes the general AC systems for the office areas, and general exhaust systems.

The alternate design replaces AHU-1 with (2) 75,000 cfm evaporative coolers, (1) 60,000 exhaust fan, and (4) 20,000 cfm relief fans. The alternate evaporative coolers employ direct



MECHANICAL SYSTEMS



evaporation only, unlike AHU-1 which is capable of automatic switching from indirect and direct, or a combination thereof depending upon the ambient weather conditions. The alternate systems have natural gas fired heaters inside the unit to provide the necessary heating during the winter months. AHU-1 is an all-inclusive ventilation unit. It has supply fans, as well as exhaust fans built into the AHU. To do an accurate equipment cost comparison, it was necessary to include the relief fans, and exhaust fans as the true total equipment cost for the alternate

	Cost of Ownership		
	Proposed System	Alternate System	Difference
Total HVAC System Cost	\$1,162,798	\$881,635	\$ 281,163
AHU Equipment Cost	\$589,620	\$392,500	\$ 197,120
Annual Electric Cost	\$52,863	\$39,988	\$ 12,875
Annual Water Cost	\$1,838	\$2,667	\$ (829)
Annual Natural Gas Cost	\$5,142	\$10,205	\$ (5,063)
Annual Maintenance Cost	\$4,688	\$7,125	\$ (2,438)
Pad Replacement Cost	\$6,000	\$8,800	\$ (2,800)
Annual Pad Replacement Cost	\$759	\$1,119	\$ (359)
Subtotals:			
Annual Utilities & Maintenance:	\$65,290	\$61,104	\$ 4,186
Grand Total:	\$1,228,088	\$942,739	\$ 285,349

	20 Year Life Cycle Cost		
	Proposed System Cost	Alternate System Cost	Total Difference
AHU Equipment Cost	\$589,620	\$392,500	\$ 197,120
Annual Utilities & Maintenance:	\$65,290	\$61,104	\$ 4,186
20 Years of Utilities & Maintenance:	\$1,305,805	\$1,222,076	\$ 83,729
Total 20 Year Cost	\$1,895,425	\$1,614,576	\$ 280,849
Grand Total:	Proposed System Cost	Alternate System Cost	Total Difference
Yearly Cost for 20 years	\$94,771.25	\$80,728.78	\$ 14,042

evaporative cooler cost. See the "cost of ownership" table above for the total cost of the alternate design system. Like the proposed system, the alternate construction cost of \$881,635 includes all the HVAC systems for the entire alternate design, minus the General Contractor markup. This includes the general AC systems for the office areas, and general exhaust systems. The last line item "Grand Total" in the "cost of ownership" table represents the total HVAC cost

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3

plus the annual utility/maintenance cost. The alternate design has the lower grand total cost, and the difference between the two is (delta \$285,349).

The next cost table is comparison between the simple 20-year cost of each system. In this comparison, only the equipment cost was used in lieu of the total HVAC cost because not all pieces of the overall system have a 20-year lifespan. The typical lifespan of AHU-1 and the large evaporative coolers will be approximately 20-years. The analysis didn't include bank loan interest, labor, material, and utility cost escalation. The analysis looked at the annual operating cost over a 20-year period plus the equipment first cost. The difference between the two systems is (delta \$14,042/year).

In summary, the alternate system has **lower first cost** (delta \$281,163) and **lower operating cost** (delta \$4,186/year) than the proposed designed system. Because of savings in both categories, there is no payback for going with the proposed design (AHU-1 system).

The proposed system was originally designed to provide maximum human comfort without paying the full price tag of a central plant (chiller & pumps), as we had done for the recently completed the Pima County Fleet Services building. The alternate system reduces the construction cost, and has minimal savings in operational cost, but it will not achieve the same level of human comfort as the proposed system.

Pima County challenged the WSM design team to provide AC like conditions for the maintenance techs, but for less cost than a traditional central cooling plant. The design team used the hybrid approach to cut the anticipated construction cost in half. Designing the hybrid Evap/DX cooling AHU, and not designing a central plant system that utilizes chillers, pumps, and chilled water piping systems, allowed Pima County to achieve their goal of maximizing human comfort for their employees, while not paying a large sum for central cooling plant.

The hybrid system out performs the standard central plant AC systems in construction cost and operational cost, but it comes up short when comparing it to traditional direct evaporative cooling systems. However, there is a value associated with human comfort. For instances, most people choose to pay higher up-front cost and operational cost to have their homes cooled by AC as opposed to evaporative cooling. Most home owners understand this value and don't mind the increased associated cost. For the Yuma Fleet Maintenance Facility that annual cost is about \$14,402.



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MECHANICAL SYSTEMS

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4

Respectfully,

PH Mechanical Engineering
P. Eric Hein, P.E., C.P.D.
Principal



Expires 9/30/19



SHOP DOORS

Review of Shop Door Options

Base bid design called for four-fold doors 16ft wide by 16ft tall. There are two of these doors required for the project and are used at the main entry and exit from the main shop space. All vehicle circulation into and out of the building with the exception of the Fire ladder truck / apparatus repair bay would use these doors. This means a significant amount of traffic daily. Estimated daily cycles for each door would be 25 or an annual total of 6500 cycles annually per door.

The other large doors into and out of the Fire ladder apparatus repair bay and into the loading dock parts area are currently specified to be sectional overhead doors due to the limited cycles required for this space and their reduced cost.

Alternatives to those doors would be Sectional Overhead doors, and Coiling Rolling overhead doors. Fabric overhead doors are not included in this review.

Coiling Rolling doors: A series of steel slats are linked to provide the door face (curtain) and roll up above the door opening when raised. A torsion spring and motor or manual chain provide the means to open the door. They are often used in loading docks at warehouses for security. Insulation is limited due the door slat construction.

The torsion spring supports the door in the open position and are rated for the number of cycles as a guideline. Standard spring cycles are rated at 20,000, a 3 year lifespan at 6500 cycles per year. Upgraded springless doors can provide up to 20 cycles per day with a 2 year warranty.

Sectional Overhead Doors

Sectional Overhead doors fold up and overhead into the building. They are typically supported by the roof structure instead of the wall like coiling doors or four fold doors. The door panels are horizontal and would be constructed of 24 ga steel skins filled with insulation.

Damage to overhead doors is often caused by drivers not waiting until the door is completely open. This makes the door unusable and the building is not secure until it is repaired. Access to the building is blocked. Estimated cost for maintenance of OH doors including replacement of sections can run about \$5,000 annually.

The torsion spring supports the door in the open position and are rated for the number of cycles as a guideline. Standard spring cycles are typically 10,000 but can be upgraded to 100,000 cycles maximum or in this project, a 15 year lifespan at 6500 cycles per year. 25 cycles a day is considered continuous use and would be

Four Fold Doors, or Bifold Doors

Door panels are constructed of welded 14 gauge steel tubular frames and 14 gauge face panels (insulated) or glazed infill panels. A factory supplied steel tube frame is anchored to the wall construction and supports the door assembly.

Standard preventative maintenance (PM) on Four-Fold doors includes applying grease to hinges and operators. With general yearly maintenance, Four-Fold door operators and components are designed and tested to perform over 1 million cycles. This represents a 40+ year lifespan. Conventional sectional and coiling doors require constant spring and cable maintenance and replacement to keep them in safe working condition. Failure to keep up with preventative maintenance on sectional and coiling steel doors can result in door failure, which can cause both damage to equipment and injury to personnel.

Four fold or bi fold doors are installed in Yuma Fire Station No. 6 and the police department and the durability and maintenance of those can be compared with the Sectional overhead doors used in the other City fire stations.



OVERHEAD DOORS

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Per City of Yuma Facilities Manager, Justin Lewis the four fold doors have 1 maintenance call for every 10 of the sectional doors. **The annual savings for the four fold doors over sectional overhead doors for 2016 was \$6,370. This equals a payback of 6.5 years at the costs below.** See appendix for further information.

Four-Fold doors clear the opening at an average speed of 24 inches per second. Conventional overhead and coiling doors operate between 8 to 12 inches per second. The high-speed operation of the Four-Fold door minimizes heating and cooling losses and reduces waiting time, increasing productivity and energy efficiency. Manual operation is simple and requires no springs under tension, chain hoists or heavy lifting.

The Four-Fold door moves horizontally rather than vertically. As a result, the door is visible throughout the opening motion allowing the driver to know the precise second the doorway is clear to exit

Cost Comparison

16x16 Fourfold Door	\$48,000
16x16 Sectional OH Door	\$ 6,300
16x16 Coiling OH Door	\$ 8,600

**APPENDIX****1. STRUCTURAL SUMMARY Pre-Engineered Building****A. Governing Design Codes and Standard****1. Design Building Codes**

- 2012 IBC.

2. Referenced Standards

- ASCE 7-10 - for load determination and application
- ACI 318-11 - for structural concrete design

B. New Construction: It is anticipated that the structural frame of the Center will be detailed as a lateral load resisting system consisting of a pre-engineered metal building (PMB), assumed with soil having a minimum bearing capacity of 3,000 lbs/SF. Elements of the construction could consist of:

- Roof live load = 20 psf (reducible)
- Collateral load = 10 psf
- Wind: V = 115 mph (3 sec gust) Ultimate, Exposure Category = C, Risk Category = II
- Seismic Design Category = D,
Site Class = D,
S_s = 0.710,
S₁ = 0.266,
S_{ds} = 0.583,
S_{d1} = 0.331,
I_e = 1.0
- Allowable bearing pressure = 3000 psf. All construction to conform to the requirements of soil report No. 016-0090 by Nicklaus Engineering, Inc.
- Typical column spacing of 25' with steel columns as part of the PMB system
- 5" thick concrete slab-on-grade over 4" granular base course at offices
- 8" thick concrete slab-on-grade over 4" granular base course at all other locations
- The minimum ultimate compressive strength of the concrete at 28 days shall be 3,000 PSI
- Foundation design will be spread footings with tie beams to resist thrust from PMB columns. Footings will be based on PMB requirements.
- All slab edges between columns will be a concrete turn down
- Masonry wainscot may occur in some locations
- Major CSI Divisions
 - a. 03 00 01; Concrete
 - b. 03 30 00; Cast-in-place Concrete
 - c. 04 22 00; Concrete Unit Masonry
 - d. 05 12 00; Structural Steel
 - e. 13 34 19; Metal Building Systems

**APPENDIX****Discussion of Four fold doors vs Sectional Overhead Doors**

From: Lewis, Justin - Facilities Maintenance Manager
 Sent: Wednesday, June 21, 2017 5:30 PM
 To: Lawrence, Timothy A. - Engineering Project Manager
 Cc: Cota, Jaime - Facilities Maintenance Supervisor
 Subject: Bi-Fold Doors

2016 Door Repair Cost

Comparison: FD Station 1 Roll up doors / FD 6 Bi-Fold Doors

Cost per year	Roll Up Doors	Bi-Fold Door	Bi-Fold Doors Savings
Repair labor cost COY	\$1,760.00	\$544.00	\$1,216.00
Contractor Invoicing Repairs	\$5,711.77	\$407.62	\$5,304.15
Parts for COY Repairs	\$300.00	\$450.00	-\$150.00
Total 2016	\$7,771.77	\$1,401.62	\$6,370.15

Hi Tim,

As we discussed today bi-fold doors have been much more reliable and require less repairs and maintenance. This is stated in the information I pasted below from one of the bi-fold door company's. For all the benefits stated below I would recommend bi-fold doors in any facility with high o/c cycles. We spend a lot of time and money maintaining and repairing the roll up doors we have now, and that means more down time. I would say it's about 10 to 1, for every 10 breakdowns on the roll up doors we only have 1 on the bi-fold doors. We have also found the bi-fold doors to be more durable, the ones at the PD have survived several vehicle collisions.

Bi-fold doors, also referred to as four-fold doors are an ideal choice when side space is available and there is a need for long lasting, low maintenance doors. With no springs to break and jamb the door, the bi-fold's design is ideal for high cycle environments such as fire stations, parking garages, and auto dealerships.

SPEED

Bi-fold doors often open faster than a conventional overhead door. Bi-folds operate at typical speeds of 24 inches per second compared to operation speeds of 8 to 12 inches per second for other door styles. A 14' x 14' bi-fold takes less than 7 seconds to open!

Faster operation has several key benefits including:

- More throughput
- Less energy loss
- Greater security

For larger door openings and vehicles, the faster speed allows more vehicles to move through the opening. Because the door closes quicker, there is less opportunity for intrusion and less air loss; an important consideration.



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APPENDIX

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HORIZONTAL MOVEMENT

The bi-fold door moves horizontally rather than vertically, making the door visible throughout the opening motion, and giving the driver a better view of the door clearance. With overhead doors drivers can misjudge the clearance available and drive into the bottom of the door as it is opening. In addition, bi-fold doors are more tolerant of dents and dings from impact in that the door will still continue to operate.

Justin Lewis

Facilities Manager, City of Yuma

One City Plaza, Yuma, AZ 85364

928.373.5121 / Fax: 928.373.5122

Justin.Lewis@YumaAZ.gov / www.YumaAZ.gov



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See Following pages for back-up data for HVAC analysis of systems.

		H.V.A.C. Original Design										Material		Labor		Total
01	1571	1599.900	Rooftop Unit, 6,000 CFM	1	Each	15000.00	15,000	40,000	40	2,000	3,060	20,060	17,700.00	2,360.00	20,060.00	
01	1571	1599.900	Rooftop Unit, 4,000 CFM	1	Each	10000.00	10,000	32,000	32	1,600	2,088	13,688	11,800.00	1,888.00	13,688.00	
01	1571	1599.900	Rooftop Unit, 3,000 CFM	1	Each	8250.00	8,250	32,000	32	1,600	1,773	11,623	9,735.00	1,888.00	11,623.00	
01	1571	1599.900	Ductless Split System	1	Each	2100.00	2,100	16,000	16	800	522	3,422	2,478.00	944.00	3,422.00	
01	1571	1599.900	Refrigerant Piping	200	Lft	8.50	1,700	0.160	32	1,600	594	3,894	10.03	9.44	19.47	
01	1571	1599.900	Refrigerant Insulation	100	Lft	7.00	700	0.000	0	0	126	826	8.26	-	8.26	
01	1571	1599.900	Indirect/Direct Air Handler w/Evap 55,000 CFM	1	Each	589620.00	589,620	120,000	120	6,000	107,212	702,832	695,751.60	7,080.00	702,831.60	
01	1571	1599.900	Evaporative Cooler, 5,000 CFM	1	Each	1500.00	1,500	12,000	12	600	378	2,478	1,770.00	708.00	2,478.00	
01	1571	1599.900	Compressor Hood	1	Each	2500.00	2,500	8,000	8	400	522	3,422	2,950.00	472.00	3,422.00	
01	1571	1599.900	Exhaust Fan, Ceiling	1	Each	200.00	200	3,000	3	150	63	413	236.00	177.00	413.00	
01	1571	1599.900	Exhaust Fan, 900 CFM	1	Each	650.00	650	8,000	8	400	189	1,239	767.00	472.00	1,239.00	
01	1571	1599.900	Exhaust Fan, 5,000 CFM In-Line	1	Each	1650.00	1,650	10,000	10	500	387	2,537	1,947.00	590.00	2,537.00	
01	1571	1599.900	Vehicle Exh Hose Reel, OFCI	14	Each	200.00	2,800	4,000	56	2,800	1,008	6,608	236.00	236.00	472.00	
01	1571	1599.900	Louver, 48x24	1	Each	350.00	350	4,000	4	200	99	649	413.00	236.00	649.00	
01	1571	1599.900	Supply Air Diffuser	25	Each	70.00	1,750	0.500	13	625	428	2,803	82.60	29.50	112.10	
01	1571	1599.900	Supply Air Diffuser, Duct Mt	76	Each	150.00	11,400	1,500	114	5,700	3,078	20,178	177.00	88.50	265.50	
01	1571	1599.900	Return Register	5	Each	70.00	350	0.500	3	125	86	561	82.60	29.50	112.10	
01	1571	1599.900	Exhaust Grille	7	Each	60.00	420	1,000	7	350	139	909	70.80	59.00	129.80	
01	1571	1599.900	Exhaust Grille, Duct Mt	21	Each	95.00	1,995	1,000	21	1,050	548	3,593	112.10	59.00	171.10	
01	1571	1599.900	Return Grille, Wall	4	Each	65.00	260	0.750	3	150	74	484	76.70	44.25	120.95	
01	1571	1599.900	Return Grille, Wall 42x36	2	Each	315.00	630	3,000	6	300	167	1,097	371.70	177.00	548.70	
01	1571	1599.900	Manual Dampers	30	Each	60.00	1,800	1,500	45	2,250	729	4,779	70.80	88.50	159.30	
01	1571	1599.900	Spin-In	27	Each	8.50	230	0.220	6	297	95	621	10.03	12.98	23.01	
01	1571	1599.900	Flex Duct	216	Lft	2.25	486	0.063	14	675	209	1,370	2.66	3.69	6.34	
01	1571	1599.900	Clamps	54	Each	4.00	216	0.000	0	0	39	255	4.72	-	4.72	
01	1571	1599.900	Flexible Connection	100	Lft	4.50	450	0.150	15	750	216	1,416	5.31	8.85	14.16	
01	1572	1593.000	Supply Air Duct, Shop	16,155	Lbs	1.95	31,502	0.053	861	43,053	13,420	87,975	2.30	3.14	5.45	
01	1574	1593.000	LPSA Duct	3,775	Lbs	1.95	7,361	0.053	201	10,060	3,136	20,558	2.30	3.14	5.45	
01	1574	1594.000	Duct Insulation	2,825	Sqft	1.50	4,238	0.000	0	0	763	5,000	1.77	-	1.77	
01	1573	1593.000	LPRA Duct	1,925	Lbs	2.95	5,679	0.053	103	5,130	1,946	12,754	3.48	3.14	6.63	
01	1576	1593.000	General Exhaust Duct	780	Lbs	1.95	1,521	0.053	42	2,079	648	4,248	2.30	3.14	5.45	
01	1576	1593.000	Exhaust Duct, Shop	16,340	Lbs	1.95	31,863	0.053	871	43,546	13,574	88,983	2.30	3.14	5.45	
01	1576	1593.000	Exhaust Duct, Tail Pipe	1,400	Lbs	1.95	2,730	0.053	75	3,731	1,163	7,624	2.30	3.14	5.45	
01	1571	1599.900	Vibration Isolation	1	Lsum	1000.00	1,000	120,000	60	3,000	720	4,720	1,180.00	3,540.00	4,720.00	
01	1571	1599.900	Commissioning	1	Lsum	2000.00	2,000	360,000	360	18,000	3,600	23,600	2,360.00	21,240.00	23,600.00	
01	1571	1596.000	Test & Balance Systems	1	Lsum	20000.00	20,000	0.000	0	0	3,600	23,600	23,600.00	-	23,600.00	
01	1571	1597.000	Controls	60	Pts	900.00	54,000	0.000	0	0	9,720	63,720	1,062.00	-	1,062.00	
01	1571	1597.000	Controls, BACnet	1	Lsum	7000.00	7,000	0.000	0	0	1,260	8,260	8,260.00	-	8,260.00	
		Total HVAC							Hours	3,190	Cost/GSF	29.89	1,162,798			

Original Hybrid design

\$589,620

		H.V.A.C. - Alternate Design													
												Material	Labor	Total	
01	1571	1599.900	Split System Heat Pump, 5 Tons	2	Each	4000.00	8,000	16,000	32	1,600	1,728	11,328	4,720.00	944.00	5,664.00
01	1571	1599.900	Split System Heat Pump, 4 Tons	2	Each	3200.00	6,400	16,000	32	1,600	1,440	9,440	3,776.00	944.00	4,720.00
01	1571	1599.900	Ductless Split System	2	Each	2100.00	4,200	16,000	32	1,600	1,044	6,844	2,478.00	944.00	3,422.00
01	1571	1599.900	Refrigerant Piping	500	Lft	8.50	4,250	0.160	80	4,000	1,485	9,735	10.03	9.44	19.47
01	1571	1599.900	Refrigerant Insulation	250	Lft	7.00	1,750	0.000	0	0	315	2,065	8.26	-	8.26
01	1571	1599.900	Evap Cooler with VFD, 75,000 CFM	2	Each	167500.00	335,000	120,000	240	12,000	62,460	409,460	197,650.00	7,080.00	204,730.00
01	1571	1599.900	Evaporative Cooler, 5,000 CFM	1	Each	1500.00	1,500	12,000	12	600	378	2,478	1,770.00	708.00	2,478.00
01	1571	1599.900	Compressor Hood	1	Each	2500.00	2,500	8,000	8	400	522	3,422	2,950.00	472.00	3,422.00
01	1571	1599.900	Exhaust Fan, Ceiling	1	Each	200.00	200	3,000	3	150	63	413	236.00	177.00	413.00
01	1571	1599.900	Exhaust Fan, 900 CFM	1	Each	650.00	650	8,000	8	400	189	1,239	767.00	472.00	1,239.00
01	1571	1599.900	Exhaust Fan, 5,000 CFM In-Line	1	Each	1650.00	1,650	10,000	10	500	387	2,537	1,947.00	590.00	2,537.00
01	1571	1599.900	Exhaust Fan, 55,000 CFM	1	Each	27500.00	27,500	24,000	24	1,200	5,166	33,866	32,450.00	1,416.00	33,866.00
01	1571	1599.900	Exhaust Fan, 25,000 CFM	4	Each	7500.00	30,000	16,000	64	3,200	5,976	39,176	8,500.00	944.00	9,794.00
01	1571	1599.900	Vehicle Exh Hose Reel, OFCI	14	Each	200.00	2,800	4,000	56	2,800	1,008	6,608	236.00	236.00	472.00
01	1571	1599.900	Louver, 48x24	1	Each	350.00	350	4,000	4	200	99	649	413.00	236.00	649.00
01	1571	1599.900	Supply Air Diffuser	25	Each	70.00	1,750	0.500	13	625	428	2,803	82.60	29.50	112.10
01	1571	1599.900	Supply Air Diffuser, Duct Mt	41	Each	150.00	6,150	1,500	62	3,075	1,661	10,886	177.00	88.50	265.50
01	1571	1599.900	Return Register	5	Each	70.00	350	0.500	3	125	86	561	82.60	29.50	112.10
01	1571	1599.900	Exhaust Grille	7	Each	60.00	420	1,000	7	350	139	909	70.80	59.00	129.80
01	1571	1599.900	Exhaust Grille, Duct Mt	21	Each	95.00	1,995	1,000	21	1,050	548	3,593	112.10	59.00	171.10
01	1571	1599.900	Return Grille, Wall	4	Each	65.00	260	0.750	3	150	74	484	76.70	44.25	120.95
01	1571	1599.900	Return Grille, Wall 42x36	2	Each	315.00	630	3,000	6	300	167	1,097	371.70	177.00	548.70
01	1571	1599.900	Manual Dampers	30	Each	60.00	1,800	1,500	45	2,250	729	4,779	70.80	88.50	159.30
01	1571	1599.900	Spin-in	27	Each	8.50	230	0.220	6	297	95	621	10.03	12.98	23.01
01	1571	1599.900	Flex Duct	216	Lft	2.25	486	0.063	14	675	209	1,370	2.66	3.69	6.34
01	1571	1599.900	Clamps	54	Each	4.00	216	0.000	0	0	39	255	4.72	-	4.72
01	1571	1599.900	Flexible Connection	100	Lft	4.50	450	0.150	15	750	216	1,416	5.31	8.85	14.16
01	1572	1593.000	Supply Air Duct, Shop	9,280	Lbs	1.95	18,096	0.053	495	24,731	7,709	50,536	2.30	3.14	5.45
01	1574	1593.000	LPSA Duct	3,775	Lbs	1.95	7,361	0.053	201	10,060	3,136	20,558	2.30	3.14	5.45
01	1574	1594.000	Duct Insulation	2,825	Sqft	1.50	4,238	0.000	0	0	763	5,000	1.77	-	1.77
01	1573	1593.000	LPRA Duct	1,925	Lbs	2.95	5,679	0.053	103	5,130	1,946	12,754	3.48	3.14	6.63
01	1576	1593.000	General Exhaust Duct	780	Lbs	1.95	1,521	0.053	42	2,079	648	4,248	2.30	3.14	5.45
01	1576	1593.000	Exhaust Duct, Shop	16,340	Lbs	1.95	31,863	0.053	871	43,546	13,574	88,983	2.30	3.14	5.45
01	1576	1593.000	Exhaust Duct, Tail Pipe	1,400	Lbs	1.95	2,730	0.053	75	3,731	1,163	7,624	2.30	3.14	5.45
01	1571	1599.900	Vibration Isolation	1	Lsum	1000.00	1,000	120,000	60	3,000	720	4,720	1,180.00	3,540.00	4,720.00
01	1571	1599.900	Commissioning	1	Lsum	2000.00	2,000	360,000	360	18,000	3,600	23,600	2,360.00	21,240.00	23,600.00
01	1571	1596.000	Test & Balance Systems	1	Lsum	20000.00	20,000	0.000	0	0	3,600	23,600	-	-	23,600.00
01	1571	1597.000	Controls	60	Pts	900.00	54,000	0.000	0	0	9,720	63,720	1,062.00	-	1,062.00
01	1571	1597.000	Controls, BACnet	1	Lsum	7000.00	7,000	0.000	0	0	1,260	8,260	8,260.00	-	8,260.00
		Total HVAC		Hours		3,003		Cost/GSF		22.66		881,635			

392,500

(197,120)

Savings from Original Design

Post Bid Alternate

This spreadsheet is a cost analysis for two different AC systems for Yuma Fleet in Yuma, Arizona. This spreadsheet will combine the annual cost to run a system along with the initial cost of it. There are a total of 10 sheets, each sheet is a different calculation. The Weather Data sheet references the annual temperature distribution over Yuma, Arizona. Some calculations that have more time restrictions have more detailed weather data attached to them.

This sheet is a summary of both systems. It is linked to the other sheets to display the calculations. The cost of each utility is in dollars

	Cost of Ownership		
	Proposed System	Alternate System	Difference
Total HVAC System Cost	\$1,162,798	\$881,635	\$ 281,163
AHU Equipment Cost	\$589,620	\$392,500	\$ 197,120
Annual Electric Cost	\$52,863	\$39,988	\$ 12,875
Annual Water Cost	\$1,838	\$2,667	\$ (829)
Annual Natural Gas Cost	\$5,142	\$10,205	\$ (5,063)
Annual Maintenance Cost	\$4,688	\$7,125	\$ (2,438)
Pad Replacement Cost	\$6,000	\$8,800	\$ (2,800)
Annual Pad Replacement Cost	\$759	\$1,119	\$ (359)
Subtotals:			
Annual Utilities & Maintenance:	\$65,290	\$61,104	\$ 4,186
Grand Total:	\$1,228,088	\$942,739	\$ 285,349

	20 Year Life Cycle Cost		
	Proposed System Cost	Alternate System Cost	Total Difference
AHU Equipment Cost	\$589,620	\$392,500	\$ 197,120
Annual Utilities & Maintenance:	\$65,290	\$61,104	\$ 4,186
20 Years of Utilities & Maintenance:	\$1,305,805	\$1,222,076	\$ 83,729
Total 20 Year Cost	\$1,895,425	\$1,614,576	\$ 280,849
Grand Total:	\$94,771.25	\$80,728.78	\$ 14,042
Yearly Cost for 20 years			

This spreadsheet displays the temperature data for Yuma, Arizona over an annual basis.
The data displays how many hours/month a dry bulb temperature is met, and what the MCWB is at that DB temperature
All weather is in degrees of Fahrenheit

Cooling Required (greater than 70°)
Heating Required (Less than 50°)

January		February		March		April		May		June		Total Cooling Hours:	
DB	MCWB	Hours	MCWB	Hours	MCWB	Hours	MCWB	Hours	MCWB	Hours	DB	Hours	5231
120											120		
115											72.8	2	115
110									70.5	1	70.2	18	110
105							67.4	1	67.7	6	69.2	56	Put April - August on DX + IEC
100							65	4	65.6	23	67.3	79	3216
95							63	1	64.1	60	66	93	100
90							60.9	7	62.4	82	64.7	97	Average Temp
85							58.2	6	60.6	68	60.9	88	90 DB(°F)
80	55.4	2	57	23	57.1	47	57.3	81	59.3	105	61.2	110	85
75	54.1	17	55	48	55	72	55.5	98	57.7	123	59.2	90	80
70	53.1	46	53.2	72	53.5	103	53.6	116	55.6	115	57.1	48	75
65	50.9	83	51.4	102	51.7	127	51.8	116	53.5	80	54.3	15	70
60	49.1	126	49.9	126	49.7	142	49.6	97	50.8	43	51.8	3	65
55	46.6	165	46.9	124	47.1	123	47.2	56	47.9	16	51	0	60
50	43.6	161	43.3	96	44.4	72	44.7	22	45.3	3		50	55
45	39.8	98	40	53	40.9	22	40.8	5				45	Average Cooling DB(°F)
40	35.6	33	36.1	17	37	3	40.2	0					Average Cooling MCWB(°F)
35	< 32	11	< 32	4	33	0							59.99
												80.31	
												40	45
												40	Total Heating Hours:
												35	421

July			August		September		October		November		December	
DB	MCWB	Hours	MCWB	Hours	MCWB	Hours	MCWB	Hours	MCWB	Hours	MCWB	Hours
120	73.5											120
115	73.5	1	74.7	1	72.5							115
110	72.9	21	74.5	14	72.7	5						110
105	72.6	79	74	70	72.8	25	69.5	4				105
100	72.3	115	73.2	112	71.9	71	68.4	13				100
95	71.9	108	72.9	111	70.2	93	66.4	31	64.5			95
90	70.9	133	72.3	135	69.5	109	64.9	63	62.2	2		90
85	69.9	156	71.6	158	69.3	131	63.6	81	60.2	16		85
80	68.3	104	69.6	102	67.4	133	32.2	102	58.3	41	56.3	80
75	63.5	22	65.6	35	63.4	89	60.6	132	56.2	64	54.6	75
70	58.1	4	60.4	6	60.2	46	58	136	54.3	91	52.6	70
65	58.3	0	58.5	0	56.4	15	54.5	101	51.8	119	50.6	65
60					53.3	2	51.2	55	49.2	147	48.5	60
55					50.7	0	47.1	19	46	129	46.8	55
50							44.1	6	42.7	77	43.2	50
45							40.3	1	38.8	27	39.3	45
40							37	0	35.8	5	35	40
35									32.3	1	31.3	35
Total Cooling Hours/Month												
Cooling Hours	743		744		702		562		214		68	
Total Hours	743		744		719		744		719		742	
Percent Cooling:	100.00		100.00		97.64		75.54		29.76		9.16	
AVG DB per cooling:	91.47		90.77		86.31		79.72		74.72		71.47	
AVG MCWB/Cooling:	70.64		71.80		68.16		56.29		56.15		53.18	

Use this to calculate electrical cost per equipment
This spreadsheet works with the Electrical Summary sheet

Inputs:
HP/Kilowatts
Hours/day (Summer & Winter)
Cents/kWHour (Summer & Winter)

Calculations for Fluctuating rate between Summer and Winter

Horse Power	Kilowatts	Hours/Day (Summer)	Hours/Day (winter)	Cents/kWHour (summer)	Cents/kWHour (Winter)
80	59.66	10	10	\$0.05	\$0.03

Summer Length	Total Summer Hours:	Summer Cost (Dollars)
6	1830	\$5,458.52

How Many Winter Months?	Total Winter Hours	Winter Cost (Dollars)
6	1818	\$3,253.64

Total Cost for 12 Months of operation is (Dollars):
\$8,712.16

Fixed Rate Calculations

Horse Power	Kilowatts	Hours/Day	Hours/Year	Cents/kWHour (fixed)
79.3	59.13	24	8760	\$0.05

Total Cost for 12 Months of operation is (Dollars):
\$25,900.69

Use this sheet to organize your final calculations.
This sheet will display the electrical loads from the supply fans, exhaust fans, and cooling apparatus for Yuma, Arizona.

Cooling hours are considered if the DB is over 70°F
Summer and Winter have different Dollars/kWHour rates
Rates are hard coded in this sheet.

Inputs:
Supply/Exhaust HP
Compressor/Pump kW

Proposed Design

The existing unit has 9 supply fans at 7.5 HP and 2 exhaust at 30 HP that all run 24/7
The unit has cooling loads that are different depending on if its summer or winter (see note 1)

Totals:	Supply *(9) 7.5 HP*	Exhaust *(2) 30 HP*	Summer Only Loads Compressor	IEC Pump	Winter Only Loads Compressor/DEC Pump	IEC Pump	Totals:
HP	67.5	60					HP
Kilowatts	50.33	44.74	88.6	1.2	79.3	0.7	1.2 Kilowatts
Summer Hours	4392	4392	3216	3216	0	0	0 Summer Hours
Summer Fee (.05 Dollars/kWHour)	\$11,054	\$9,825	\$14,247	\$193	0	0	0 Summer Fee (.05 Dollars/kWHour)
Winter Hours	4363	4363	0	0	2015	2015	2015 Winter Hours
Winter Fee (.03 Dollars/kWHour)	6588	5856	0	0	\$4,794	\$42	\$73 Winter Fee (.03 Dollars/kWHour)
Total Hours	8755	8755	3216	3216	2015	2015	Total Hours
Cost/Year (dollars)	\$17,642	\$15,681	\$14,439	\$193	\$4,794	\$42	\$73 Cost/Year (dollars)

Existing Unit
Final Cost per Year (dollars)
\$52,863

NOTE 1
There are 5231 Hours of Cooling/Year During months April-August the DX + IEC is operating. (3216 Hours, Summer Rate) The rest of the cooling hours are put on the IEC + DEC (2015 Hours, Winter Rate) All hour calculations can be found on the Weather Data sheet

Alternate Design

The alternate is (2) 40 HP VFD units. During occupied hours (7AM-6PM) units operate at 100%, other hours the units operate at 60% (24 HP each)
There are (2) exhaust systems and (2) pumps (see note 2)

Supply Fan HP *(2) 40 HP VFD*	80	48	Totals:
Kilowatts	59.66	35.79	Kilowatts
Summer Hours (11/Day, 13/Day)	2013	2379	4392 Summer Hours (11/Day, 13/Day)
Summer Fee (.05 Dollars/kWHour)	\$6,004	\$4,258	\$10,262 Summer Fee (.05 Dollars/kWHour)
Winter Hours (11/Day, 13/Day)	2000	2363	4363 Winter Hours (11/Day, 13/Day)
Winter Fee (.03 Dollars/kWHour)	\$3,579	\$2,537	\$6,117 Winter Fee (.03 Dollars/kWHour)
Cost (Dollars)	\$9,584	\$6,795	\$16,379 Cost (Dollars)
Total Cost of Supply:	\$16,379		Total Cost of Supply

Exhaust Fan HP *(1) 75.8 HP (4) 7.5 HP*	75.8	30	Totals:
Kilowatts	56.52	22.37	Kilowatts
Summer Hours (24/7, 10/Day)	4392	1830	6222 Summer Hours (11/Day, 13/Day)
Summer Fee (.05 Dollars/kWHour)	\$12,413	\$2,047	\$14,460 Summer Fee (.05 Dollars/kWHour)
Winter Hours (24/7, 10/Day)	4363	1818	6181 Winter Hours (11/Day, 13/Day)
Winter Fee (.03 Dollars/kWHour)	\$7,398	\$1,220	\$8,619 Winter Fee (.03 Dollars/kWHour)
Cost (Dollars)	\$19,811	\$3,267	\$23,078 Cost (Dollars)
Total Cost of Exhaust:	\$23,078		Total Cost of Exhaust

Pump *(2) 1.2 kW*	
Kilowatts	2.40
Summer Hours (24/7)	3,216
Summer Fee (.05 Dollars/kWHour)	\$386
Winter Hours (24/7)	2015
Winter Fee (.03 Dollars/kWHour)	\$145
Total Cost (Dollars)	\$531

Alternate Units
Final Cost per Year (dollars)
\$39,988

NOTE 2
The supply fans run 11 Hours/day at 80 HP, and for 13 Hours/day at 48 HP, all year The 75.8 HP exhaust fan runs 24/7 and the (4) 7.5 HP fans run 10 hours/day all year The pumps (2) run 5231 hours/year for cooling.

Alternative System Water Cakes

This spreadsheet will calculate how many gallons of water the unit uses

DB = Dry Bulb Temperature, MCVB = Mean Wet Bulb Temperature

MCVB = Temperature Gradient between DB and MCVB

GPH (Gallons Per Hour) = (CFM*EFF*DeltaT)/8700

Gallons = GPH * Hours Ran

Dump Reservoir Twice A Day

Gallons Evaporated

87319

Gallons Dumped (730 Times/Year)

172013

Daytime Constants:		Universal	
EFF	70%	1 Unit	748 Gallons
CFM	140000	0.0.00 Units	1.5 CFM
GPH Constant	8700	1.5 GPH	70000
Nighttime Constants:		EFF	
DB	70	1.5 GPH	70000
MCVB	53.1	1.5 GPH	70000
Delta	16.9	1.5 GPH	70000
Total Gallons used		4700	

Total Gallons used	1045531
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Total Units	3198
-------------	------

Total Cost (Dollar/Year)	\$2,607
--------------------------	---------

Quantities of Each Unit		
[0,10]	[11,30]	[31,∞]
120	351	927
Cost per Section		
\$ 180.00	\$ 614.25	\$ 1,872.54

This spreadsheet will calculate the cost of natural gas for the proposed unit:
 BTU/Hour = CFM * 1.08 * Delta
 CFM = Cubic Feet per Minute
 Delta = Dry bulb temperature difference across the coil

Constants:	
CFM	55000
Sens. Const.	1.08
1 Therm	100000 BTU
1 Therm	\$0.57117
Delta	30

Note:

- 1) The Delta T across the heating coil is a constant 30°F
- 2) Add an extra 20% of required BTU/Hour to total to account for Efficiency

BTU/Hour	Extra 20% For Eff	Hours of Heating/Year	BTU/Year	Therms/Year	Cost/Year
1782000	2138400	421	900266400	9003	\$5,142

Jan				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	69	13	16	
40	27	3	3	
35	9	1	0	
30	1	0	0	

Feb				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	41	6	6	
40	15	1	1	
35	4	0	0	
30	0	0	0	

March				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	21	0	1	
40	3	0	0	
35	0	0	0	
30	0	0	0	

November				
DB°	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	23	1	3	
40	5	0	0	
35	1	0	0	
30	0	0	0	

December				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	73	12	16	
40	28	3	4	
35	8	0	1	
30	2	0	0	

Total Hours				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 PM	Total Hours
45	227	32	42	301
40	78	7	8	93
35	22	1	1	24
30	3	0	0	3

This spreadsheet will calculate the cost of natural gas for the alternate unit.
 BTU/Hour = CFM * 1.08 * Delta
 CFM = Cubic Feet per Minute
 Delta = Dry bulb temperature difference across the coil

Note:
 1) There are (2) units operating at 35,000 CFM each
 2) Delta is **not** constant, air is heated to 90°F
 3) Add an extra 20% of required BTU/Hour to total to account for Efficiency

Constants:	
CFM	70000
Sens. Const.	1.08
1 Therm	100000 BTU
1 Therm	\$0.57117

DB°F	Delta°F	BTU/Hour	Extra 20%	Hours Operated	BTU/Year	Total BTU/Year	Therms/Year	Cost/Year
45	45	3402000	4082400	301	1228802400			
40	50	3780000	4536000	93	421848000	1786730400	17867	\$10,205
35	55	4158000	4989600	24	119750400			
30	60	4536000	5443200	3	16329600			

Jan				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	69	13	16	
40	27	3	3	
35	9	1	0	
30	1	0	0	

Feb				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	41	6	6	
40	15	1	1	
35	4	0	0	
30	0	0	0	

March				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	21	0	1	
40	3	0	0	
35	0	0	0	
30	0	0	0	

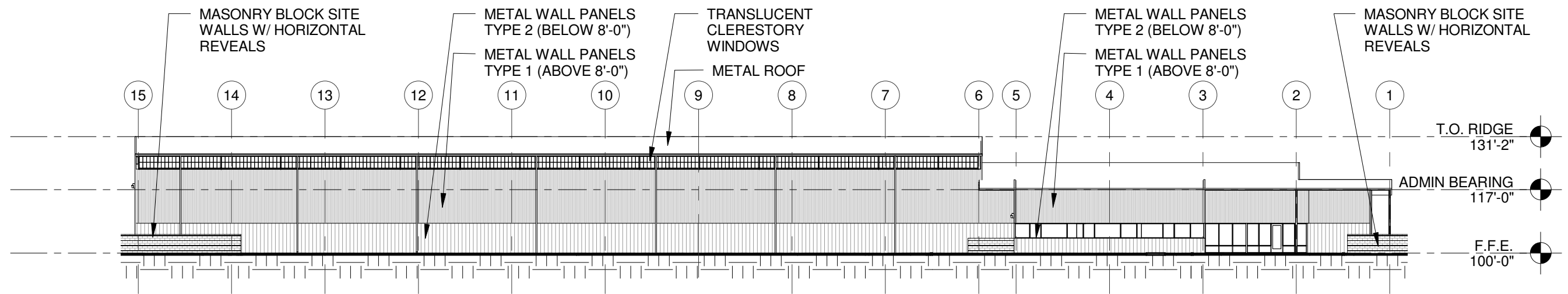
November				
DB°	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	23	1	3	
40	5	0	0	
35	1	0	0	
30	0	0	0	

December				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	
45	73	12	16	
40	28	3	4	
35	8	0	1	
30	2	0	0	

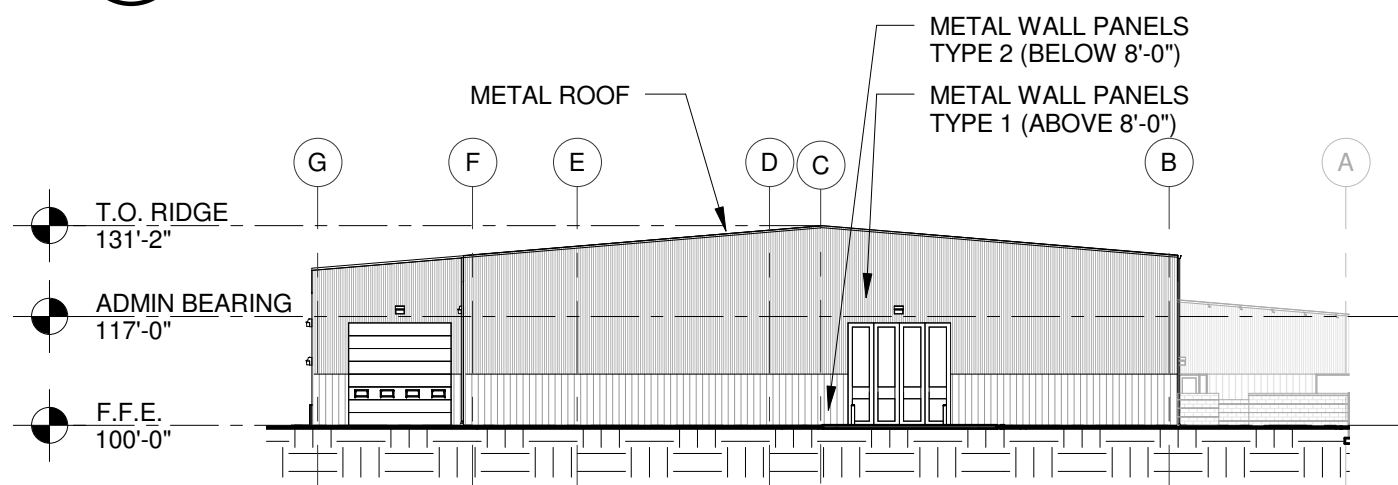
Total Hours				
DB°F	1 AM - 8 AM	9 AM - 4 PM	5 PM - 12 AM	Total Hours
45	227	32	42	301
40	78	7	8	93
35	22	1	1	24
30	3	0	0	3

Proposed System Maintenance:		Maintenance rate:	
		\$	75.00
This is the annual cost of maintenance for the proposed unit			
Rate of maintenance is \$75/hour			
Replace one pad every 8 years at \$6,000			
Description	Occurrence/Year	Time (hours)	Annual Cost
1) Fan Equipment + Accessories			
Visual Inspections	2	0.5	\$75
Grease fan bearing	2	1	\$150
Adjust Fan Wheel	2	1	\$150
Grease slide rail bolts/check motor	2	1	\$150
Adjust Fan Belt	12	1	\$900
Adjust Drive Alignment	1	1	\$75
Adjust Propeller	2	1	\$150
Grease Motor Hold Down	2	1	\$150
Coolers			
Visual Inspections	2	1	\$150
Sump Cleaning	12	2	\$1,800
Pad Replacement over 8 years (6000 for pad)	0.125	1	\$759
3) Misc. Inspections/Maintenance			
Visually Inspect/adjust if needed:			
Outside air inlet Louvers/Dampers	2	1	\$150
Filters/filter system	2	1	\$150
Water overflow connection	2	1	\$150
4) Initial Start up			
Clean System/Flush Water	1	1	\$75
Check/Adjust: Fans, Belt Tension	1	1	\$75
Check/Adjust: Float, Water Distribution	1	1	\$75
Check/Adjust Locking Collar + Fan Wheel	1	1	\$75
5) Seasonal Shutdown			
Drain/Clean Cold Water Sump	1	1	\$75
Grease Fan/Motor bearings	1	1	\$75
Final inspection	1	0.5	\$38
		Total Cost w/o pads:	
		\$4,688	

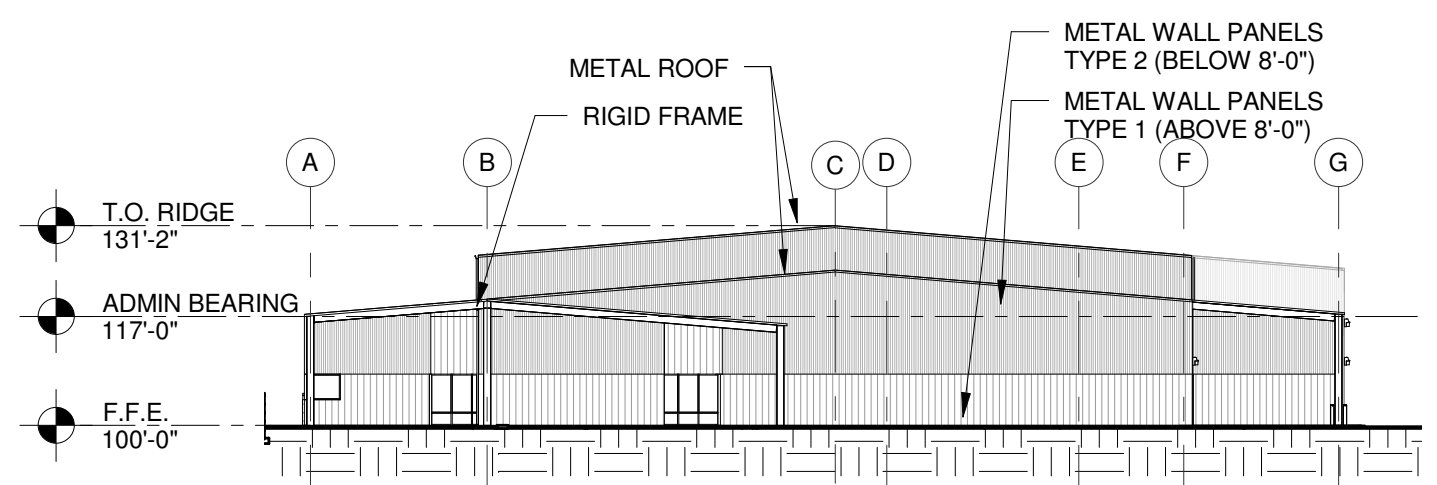
Alternative System Maintenance:		Maintenance rate:	
		\$	75.00
This is the annual maintenance cost for two units.			
Rate of maintenance is \$75/hour			
Replace 2 pads every 8 years for \$8,800			
Description	Occurrence/Year	Time (hours)	Annual Cost
1) Fan Equipment + Accessories			
Visual Inspections	2	1	\$ 150
Grease fan bearing	2	2	\$ 300
Adjust Fan Wheel	2	2	\$ 300
Grease slide rail bolts/check motor	2	2	\$ 300
Adjust Fan Belt	12	2	\$ 1,800
Adjust Drive Alignment	1	2	\$ 150
Adjust Propeller	2	2	\$ 300
Grease Motor Hold Down	2	2	\$ 300
2) Evap Coolers			
Visual Inspections	2	2	\$ 300
Sump Cleaning	12	2	\$ 1,800
Pad Replacement over 8 years (8,800 for pads)	0.125	2	\$ 1,119
3) Misc. Inspections/Maintenance			
Visually Inspect/adjust if needed:			
Outside air inlet Louvers/Dampers	2	1	\$ 150
Filters/filter system	2	1	\$ 150
Water overflow connection	2	1	\$ 150
4) Initial Start up			
Clean System/Flush Water	1	2	\$ 150
Check/Adjust: Fans, Belt Tension	1	2	\$ 150
Check/Adjust: Float, Water Distribution	1	2	\$ 150
Check/Adjust Locking Collar + Fan Wheel	1	2	\$ 150
5) Seasonal Shutdown			
Drain/Clean Cold Water Sump	1	2	\$ 150
Grease Fan/Motor bearings	1	2	\$ 150
Final inspection	1	1	\$ 75
		Total Cost w/o pads:	
		\$	7,125



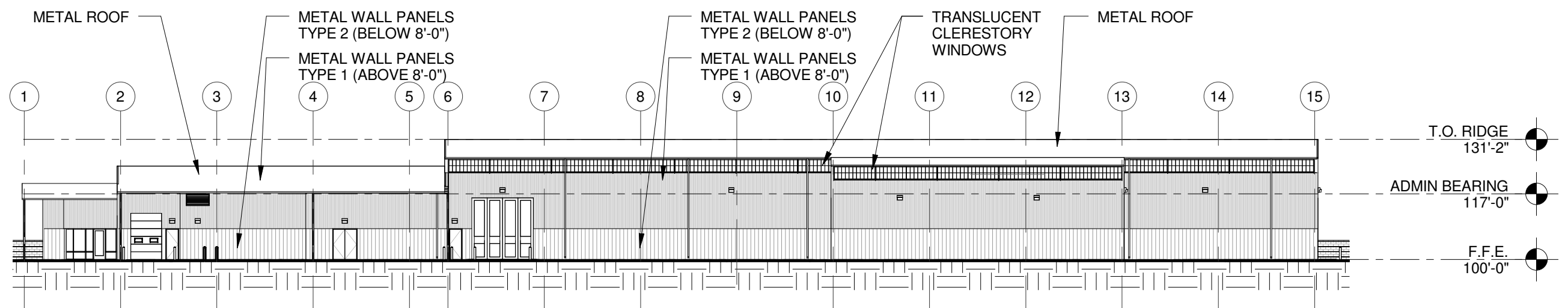
1 NORTH ELEVATION
1" = 30'-0"



2 EAST ELEVATION
1" = 30'-0"

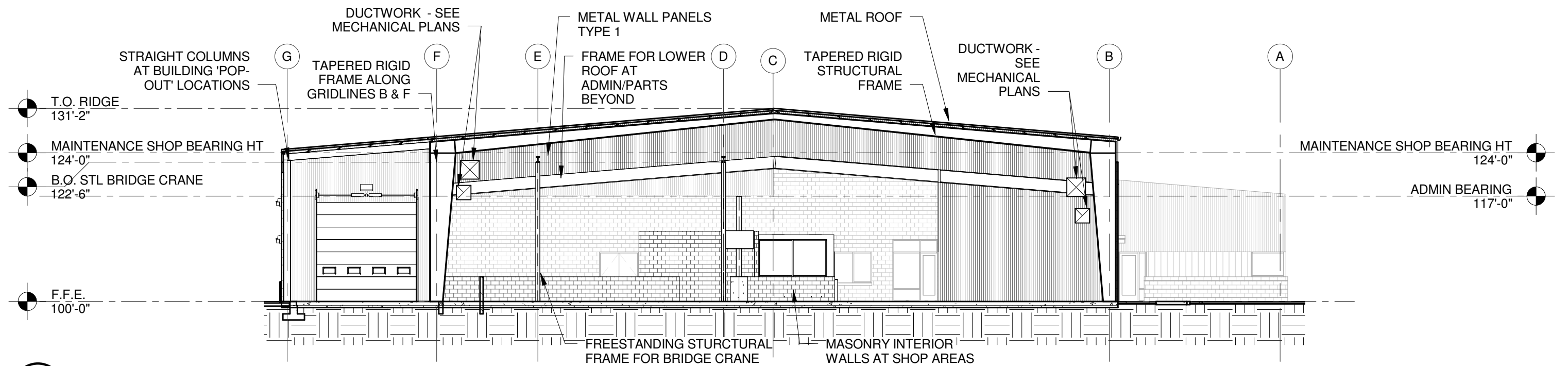


3 WEST ELEVATION
1" = 30'-0"



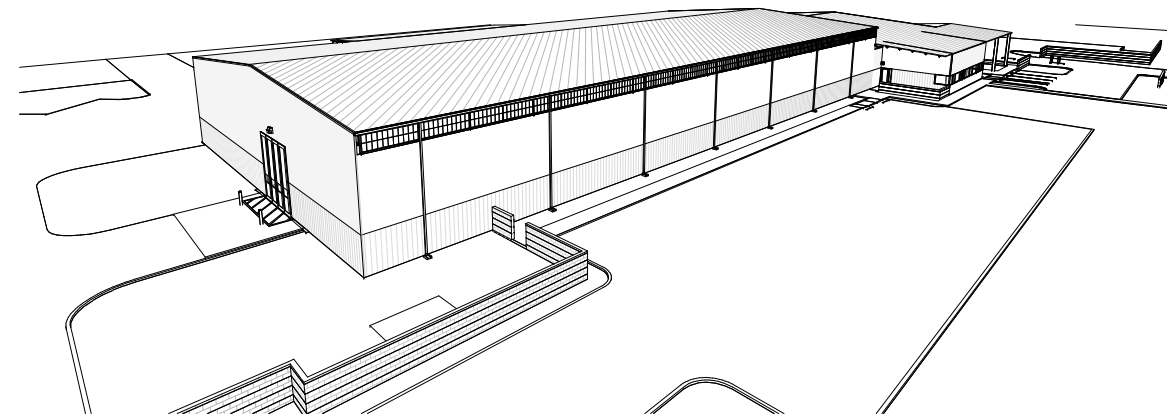
4 SOUTH ELEVATION
1" = 30'-0"

METAL BUILDING OPTION

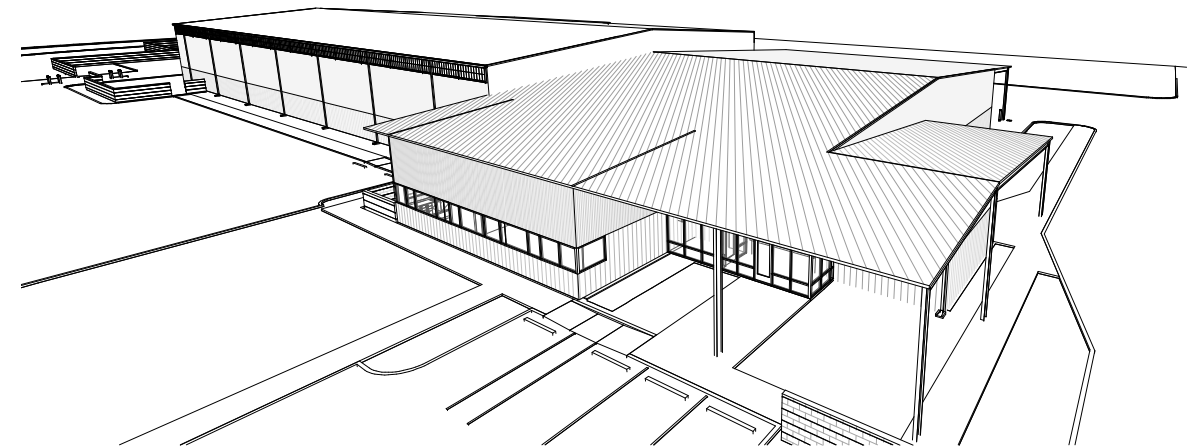


1 MAINTENANCE BAY SECTION

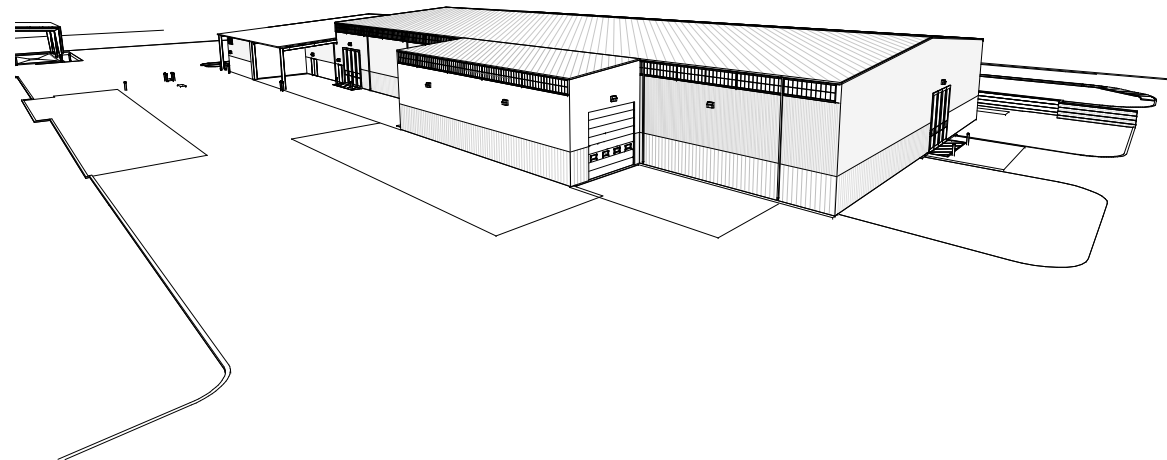
1/16" = 1'-0"



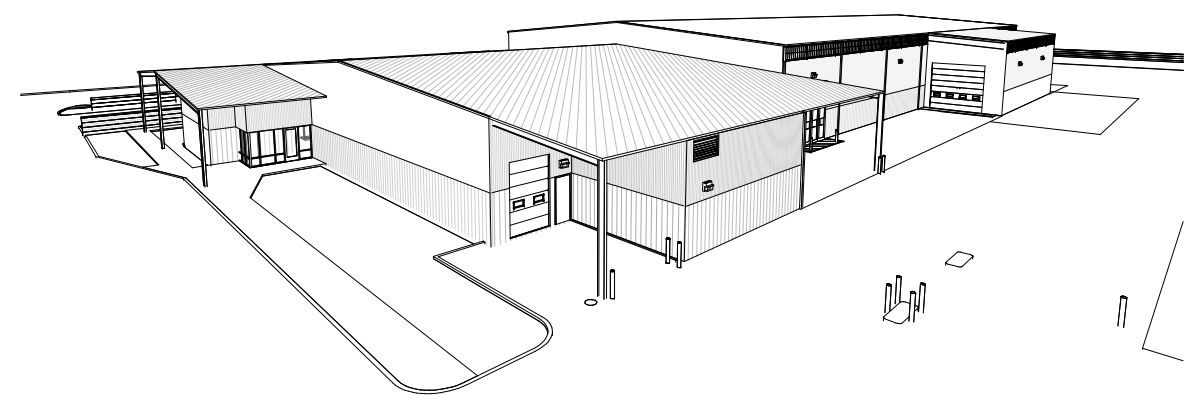
VIEW FROM NORTHEAST



VIEW FROM NORTHWEST



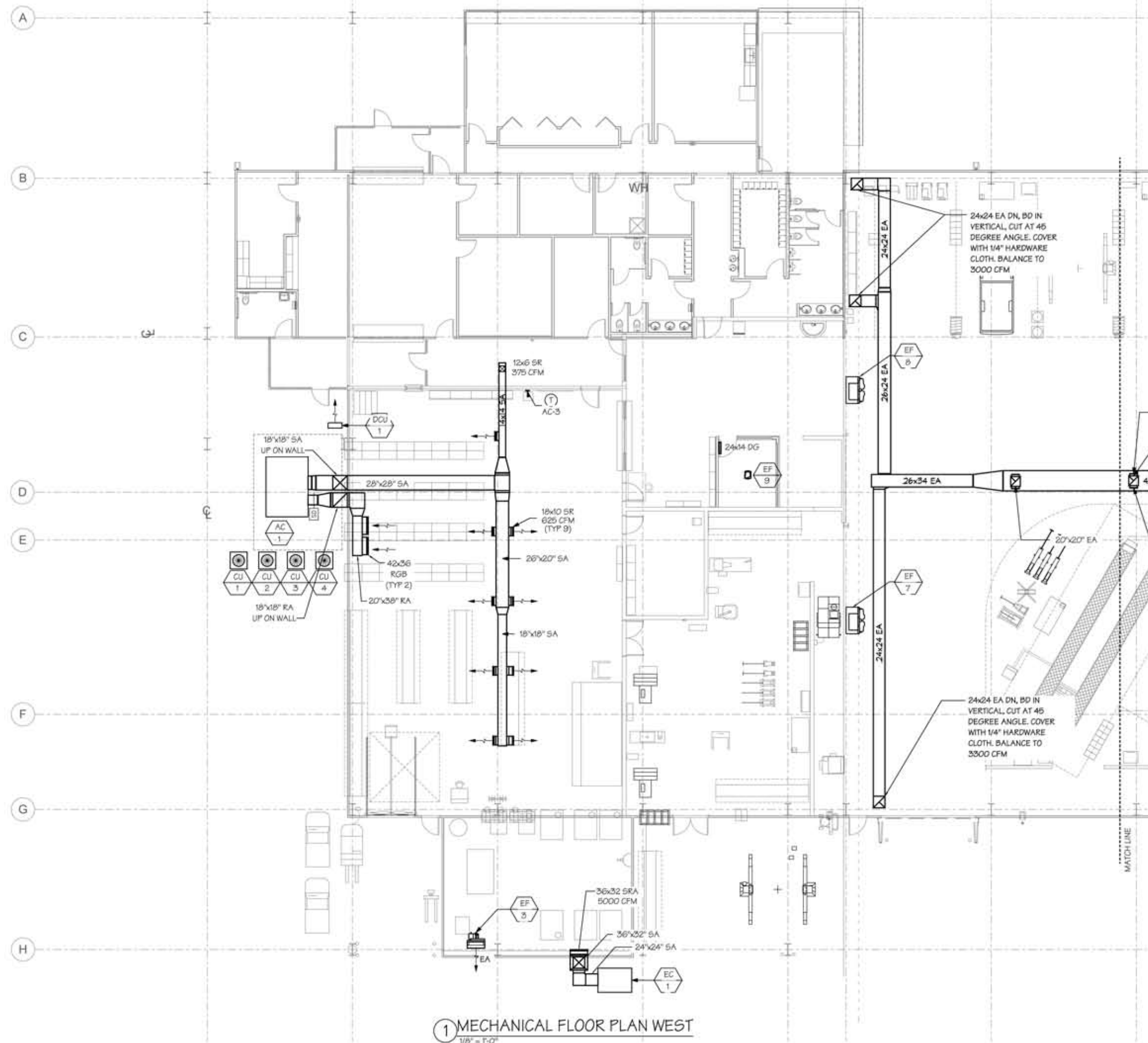
VIEW FROM SOUTHEAST



VIEW FROM SOUTHWEST

YUMA FLEET MAINTENANCE FACILITY

PRINCIPLES



1 MECHANICAL FLOOR PLAN WEST
1/8" = 1'-0"

REVISIONS

C.O.Y. COMMENTS 2/3/17

ADDENDUM #2 3/29/17

PROJECT NO.:

DEATH

IAS

CHECKED

DATE _____

PLOT DATA

SHEET 11

MECHANICAL
FLOOR PLAN
WEST

SHEET NO.

• • • • •

1000 1000 1000 1000 1000 1000

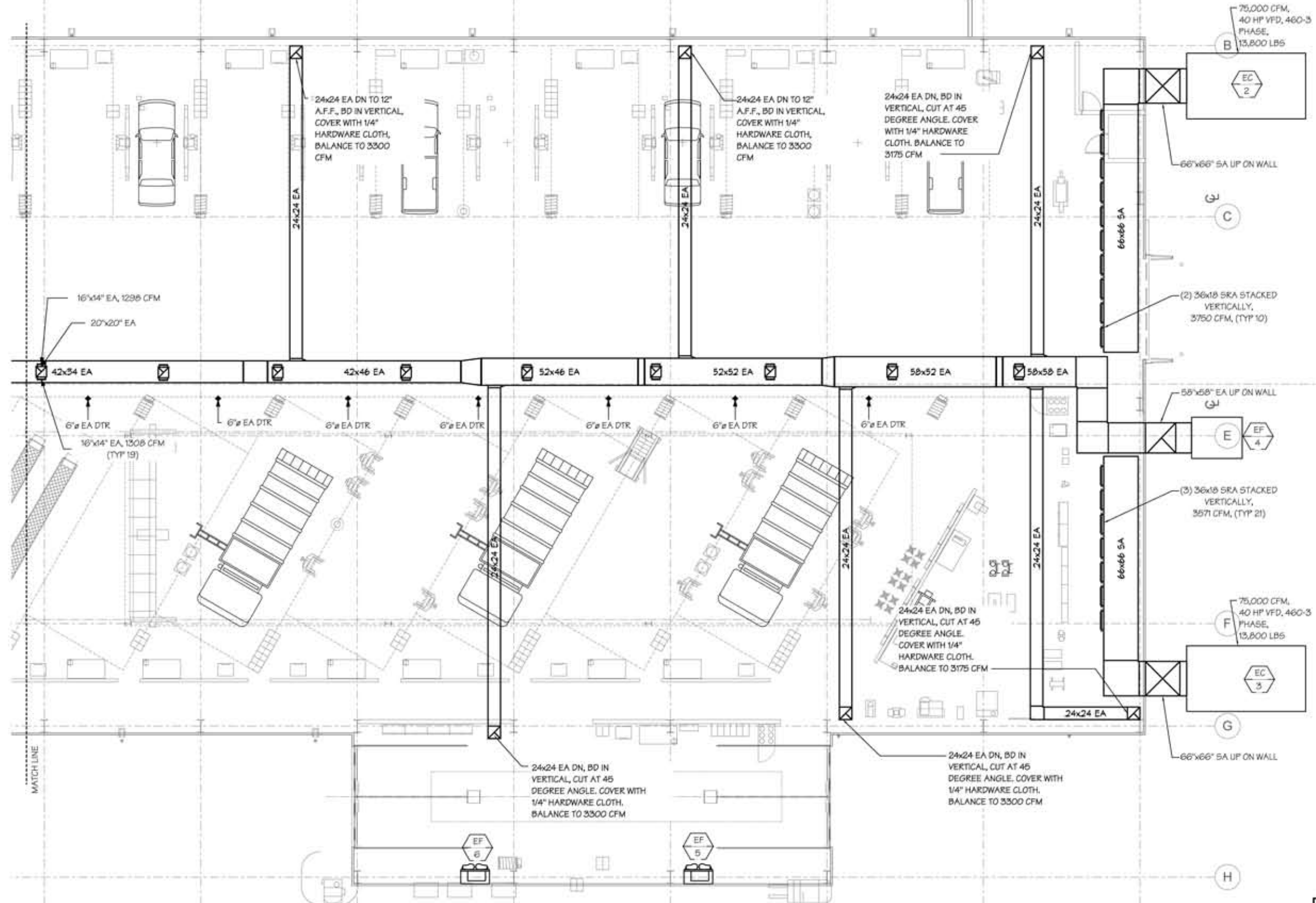
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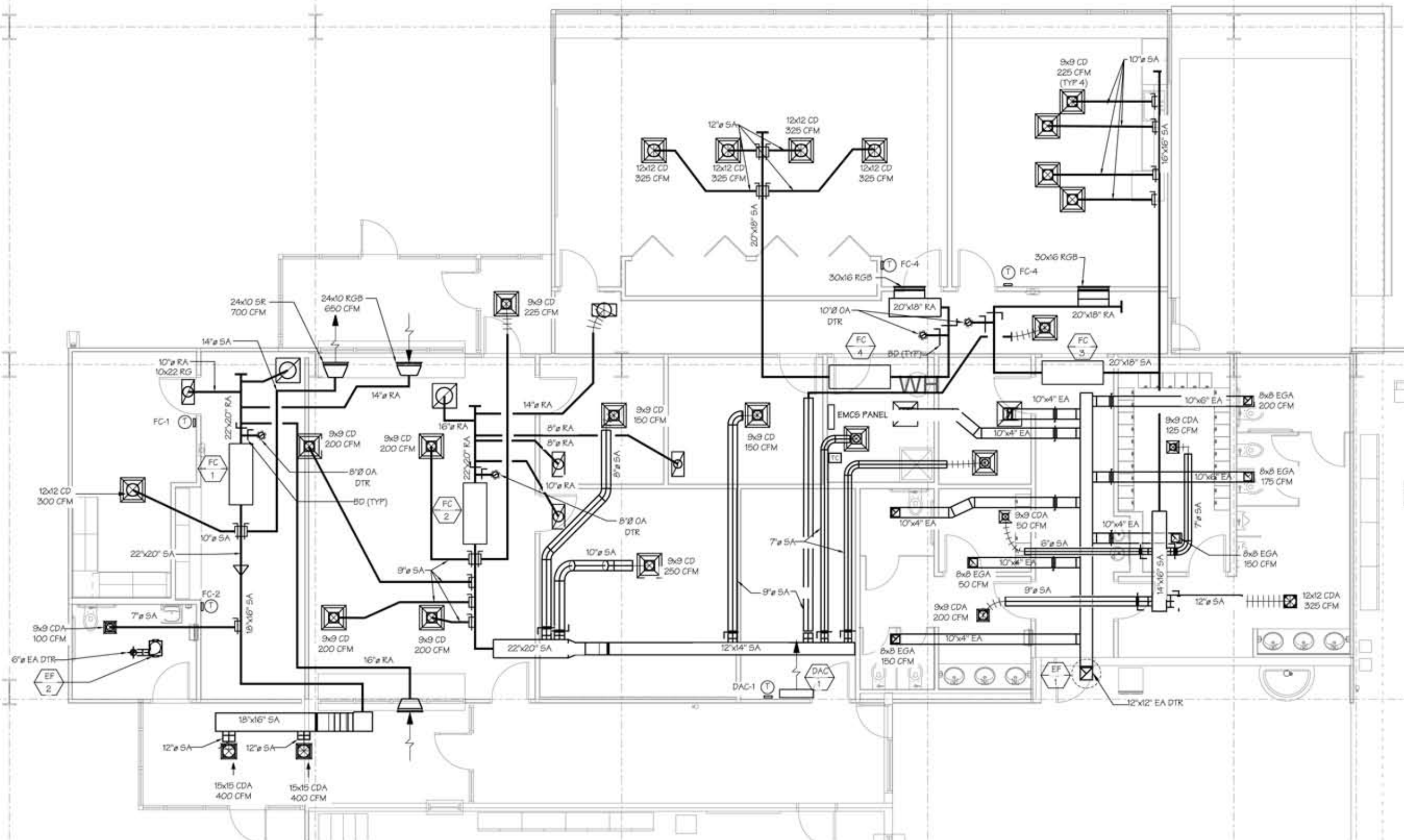
• 44100 •

M100

• **M100** •



① MECHANICAL FLOOR PLAN EAST
1/8" = 1'-0"



1 MECHANICAL FLOOR PLAN-ENLARGED OFFICE AREA
1/8" = 1'-0"

REVISIONS:

	C.O.Y. COMMENTS	2/3/17
1	ADDENDUM #2	3/29/17
2		
3		
4		
5		
6		
7		
8		
9		
10		

PROJECT NO.:

DRAWNE LAS

Checkride:

DATE:

PLOT DATE:

Sheet title:

MECHANICAL
ENLARGED
OFFICE AREA

SHEET NO. 2

• • • • •

• • • • •

M102

• **WITNESS** •

EXHAUST FAN SCHEDULE

MARK	SERVES	MFR AND MODEL	① TYPE	CFM	ESP IN WG	FAN RPM	MOTOR DATA				DISCONNECT	BACK DRAFT DAMPER	SONES	WEIGHT	REMARKS
							SPEED	WATTS	HP	ELECTRIC					
EF-1	RESTROOMS-LOCKERS	COOK 120 ACEB	ROOF MOUNTED	900	0.5	1669	1	--	1/4	115-1-60	YES	YES	1.5	15	3 4 5
EF-2	RESTROOM	COOK GC 148	CEILING MOUNTED	100	0.5	1075	1	42.0	--	115-1-60	YES	YES	2.5	25	2 4 5
EF-3	COMPRESSOR	COOK 30XMM	WALL MOUNTED	5000	0.3	717	1	--	3/4	115-1-60	YES	YES	17.0	175	2 4 6 1
EF-4	VEHICLE BAY	COOK CF SPSI	UTILITY SET, UPBLAST	55,000	0.5	956	1	--	75.8	460-3-60	YES	YES	-	5300	2 4 5
EF-5	VEHICLE BAY	COOK XLNH	WALL MOUNTED PROP FAN	25,000	0.5	582	1	--	7.5	460-3-60	YES	YES	38	775	2 4 5 1
EF-6	VEHICLE BAY	COOK XLNH	WALL MOUNTED PROP FAN	25,000	0.5	582	1	--	7.5	460-3-60	YES	YES	38	775	2 4 5 1
EF-7	VEHICLE BAY	COOK XLNH	WALL MOUNTED PROP FAN	25,000	0.5	582	1	--	7.5	460-3-60	YES	YES	38	775	2 4 5 1
EF-8	VEHICLE BAY	COOK XLNH	WALL MOUNTED PROP FAN	25,000	0.5	582	1	--	7.5	460-3-60	YES	YES	38	775	2 4 5 1
EF-9	OFFICE	COOK GC 542	CEILING MOUNTED	300	0.3	1600	1	121	--	115-1-60	YES	YES	5.5	30	5

1. BASIS OF DESIGN.
2. PROVIDE MANUFACTURER'S HANGING ISOLATION KIT, ALUMINUM GRILLE.
3. PROVIDE MANUFACTURER'S ROOF CURB AND DISCONNECT.
4. INTERLOCKING, WIRING, & CONDUIT PER ELECTRICAL DIVISION SPECIFICATIONS. PROVIDE INTERLOCK RELAYS & CONTACTS PER ELECTRICAL DRAWINGS.
5. INTERLOCK WITH TIME CLOCK, FAN TO OPERATE DURING OCCUPIED HOURS.
6. INTERLOCK WITH EG-1.
7. PROVIDE MANUFACTURER'S WIRE GUARD, BAROMETRIC BACKDRAFT DAMPER, AND DISCONNECT SWITCH.

PACKAGE ROOFTOP AIR CONDITIONING UNIT SCHEDULE

								COOLING CAPACITY AT CONDITIONS (110/80/66)			② HEATING CAPACITY		ELECTRICAL DATA								
MARK	NOM TONS	MFR	MODEL	DISCH	CFM	① OA	ESP	TOTAL MBH	SENS MBH	IEER	TYPE	TOTAL INPUT MBH	ELECTRICAL	MAX HP	MCA	MOCP	MFG'S CURB HEIGHT	FILTERS	UNIT WEIGHT	DIMENSIONS HxLxW	REMARKS
AC-1	15	TRANE	GAC-180	SIDE	6000	SEE OA SCHED	0.5	161.3	125.3	12.8	GAS	240/192	208/3/60	3	79	100	14"	2" PLEATED	1864	60x124x81	③ ④ ⑤ ⑥ ⑦ ⑧

1. SET MINIMUM O.A. DAMPER STOP TO MINIMUM O.A. REQUIREMENT.
2. HEATING CAPACITY AT STANDARD CONDITIONS.
3. SET THE SUPPLY FAN ON DURING OCCUPIED HOURS THROUGH THE PROGRAMMABLE THERMOSTAT.
4. PROVIDE LOCKING 7-DAY PROGRAMMABLE THERMOSTAT. TSTAT SHALL HAVE MIN. 5° SET POINT OVERLAP RESTRICTION, AND OFF-HOUR CONTROLS CAPABLE OF AUTOMATIC STARTUP, AUTOMATIC SETBACK & SHUTDOWN, AND 2 HOUR OVERRIDE. THERMOSTAT SHALL COMPLY WITH THE 2012 IECC SECTIONS C403.2.4.1, C403.2.4.2 & C403.2.4.3.
5. PROVIDE MEDIUM STATIC, BELT DRIVE MOTOR.
6. PROVIDE FACTORY INSTALLED ECONOMIZER WITH A DRY BULB SENSOR AND BAROMETRIC RELIEF DAMPER CAPABLE OF RELIEVING UP TO 100% RETURN AIR.
7. ELECTRICAL CONTRACTOR TO PROVIDE AND WIRE SMOKE DETECTOR, MECHANICAL CONTRACTOR TO INSTALL IN DUCTWORK.
8. PROVIDE SEISMIC ROOF CURB.

HEAT PUMP SPLIT SYSTEM SCHEDULE

INDOOR FAN COIL UNIT																	OUTDOOR UNIT										
					COOLING CAPACITY AT (104/75/61)			HEATING CAPACITY @ 65° EDB 21° OA			ELECTRICAL DATA							ELECTRICAL									
MARK	MFR AND MODEL	DISCH	CFM	MIN OA CFM	ESP TWG	TOTAL MBH	SENS MBH	SEER	MBH	ELEC HEAT KW	HSPF	ELECTRICAL	MAX HP	MCA	MOCP	DIMENSIONS LxWxH	LBS	MARK	NOM TONS	MFG AND MODEL	V/PH/Hz	MCA	MOCP	DIMS	LBS	REMARKS	
FC-1	CARRIER FV4CNB006	HORIZ	2000	6	0.5	52.2	41.6	16.0	43.3	NA	9.0	208/1/60	3/4	8.5	15	60x23x25	225	CU-1	5	CARRIER 25HCB660	208-1-60	31.5	60	35x35x45	350	1 2 3 4 5 7	
FC-2	CARRIER FV4CNB006	HORIZ	2000	6	0.5	52.2	41.6	16.0	43.3	NA	9.0	208/1/60	3/4	8.5	15	60x23x25	225	CU-2	5	CARRIER 25HCB660	208-1-60	31.5	60	35x35x45	350	1 2 3 4 5 7	
FC-3	CARRIER FV4CNB005	HORIZ	1600	6	0.5	44.8	34.6	16.0	47.4	NA	9.0	208/1/60	1/2	5.4	15	54x23x22	200	CU-3	4	CARRIER 25HCB648	208-1-60	29.8	50	35x35x40	300	1 2 3 4 5 7	
FC-4	CARRIER FV4CNB005	HORIZ	1600	6	0.5	44.8	34.6	16.0	47.4	NA	9.0	208/1/60	1/2	5.4	15	54x23x22	200	CU-4	4	CARRIER 25HCB648	208-1-60	29.8	50	35x35x40	300	1 2 3 4 5 7	

1. ELECTRICAL CONTRACTOR TO PROVIDE SEPARATE POWER POINT CONNECTIONS TO FAN COIL UNIT AND OUTDOOR UNIT.
2. PROVIDE ALL FEATURES STANDARD TO THE UNIT SCHEDULED.
3. PROVIDE LOW VOLTAGE CONTROL POWER TRANSFORMER, FAN RELAY, LIQUID LINE FILTER DRYER, AND ANTI-CYCLING CONTROL TO PREVENT RAPID COMPRESSOR CYCLING.
4. PROVIDE LOCKING 7-DAY PROGRAMMABLE, AUTOMATIC CHANGEOVER, HEAT PUMP THERMOSTAT. TSTAT SHALL HAVE SUPPLEMENTARY HEAT CONTROL (IF REQUIRED); MIN. 5° SET POINT OVERLAP RESTRICTION, AND OFF-HOUR CONTROLS CAPABLE OF AUTOMATIC STARTUP, AUTOMATIC SETBACK & SHUTDOWN, AND 2 HOUR OVERRIDE. THERMOSTAT SHALL COMPLY WITH THE 2012 IECC SECTIONS C403.2.4.1, C403.2.4.2 & C403.2.4.3.
5. SET THE SUPPLY FAN ON DURING OCCUPIED HOURS THROUGH THE PROGRAMMABLE THERMOSTAT.
6. SET MINIMUM OUTSIDE AIR DAMPER STOP TO THE MINIMUM OUTSIDE AIR REQUIREMENT. SEE OA SCHEDULE FOR AMOUNT.
7. FAN COIL SHALL HAVE FACTORY INSTALLED SINGLE POINT POWER CONNECTION.

DUCTLESS COOLING ONLY SPLIT SYSTEM SCHEDULE

INDOOR UNIT (FAN COIL)										OUTDOOR UNIT (HEAT PUMP)									
MARK	SERVES	MFR/ MODEL	NOM TONS	DISCHARGE/ TYPE	CFM- HIGH/MED/LOW	COOLING CAP. MBH	ELECTRICAL	LxWxH (IN)	LBS	MARK	MFR/ MODEL	SEER	ELECTRICAL	MCA	MFA	FLA	LxWxH (IN)	LBS	REMARKS
DAC-1	MDF	SANYO KS1271	1	WALL MOUNTED	212/259/294	11.9	115/1/60	33x8x12	20	DCU-1	SANYO C1271	17	115/1/60	10.9	20	0.35	29x11x22	100	1 2 3 4 5 6 7 8 9
DAC-2	DATA	SANYO KS1271	1	WALL MOUNTED	212/259/294	11.9	115/1/60	33x8x12	20	DCU-2	SANYO C1271	17	115/1/60	10.9	20	0.35	29x11x22	100	1 2 3 4 5 6 7 8 9

1. BASIS OF DESIGN OR EQUAL.
2. PROVIDE LOW AMBIENT CONTROL TO 0° F.
3. PROVIDE WALL MOUNTING KIT.
4. PROVIDE 24V WIRED WALL CONTROLLER (THERMOSTAT) W/ ADAPTER AS REQUIRED.
5. PROVIDE INTEGRAL FACTORY CONDENSATE PUMP, SEE PLUMBING PLANS FOR CONDENSATE ROUTING AND TERMINATION.
6. PROVIDE ALL FEATURES STANDARD TO THE UNIT SCHEDULED.
7. INSULATE BOTH GAS AND LIQUID LINES W/ ARMI-FLEX PIPE INSULATION.
8. ELECTRICAL CONTRACTOR TO SINGLE POWER POINT CONNECTION TO OUTDOOR UNIT, INDOOR UNIT TO BE SUB-FED BY OUTDOOR UNIT.
9. UNIT SHALL BE SET FOR 24/7 COOLING.

EVAPORATIVE COOLER SCHEDULE

MARK	SERVES	MFR	MODEL	DISCHARGE	CFM	ESP.	SPEED	HP	ELECTRIC	OPERATING WEIGHT	DIMS LxWxH	REMARKS
EC-1	COMPRESSOR ROOM	AEROCOOL	15500	SIDE	5000	0.5	1	3/4	115-1-60	575	70x50x52	1 2 3 4 5
EC-1	VEHICLE BAY	MESSCO	CUSTOM	SIDE	75,000		VFD	40	460-3-60	13,800	126x228x122	1 2 4 6
EC-1	VEHICLE BAY	MESSCO	CUSTOM	SIDE	75,000		VFD	40	460-3-60	13,800	126x228x122	1 2 4 6

1. UNIT COMPLETE WITH MEDIA PADS & 115 VOLT CLEAN MACHINE PUMP.
2. PROVIDE BELTS AND SHEAVES AS REQUIRED TO BALANCE TO CFM SHOWN.
3. PROVIDE FOUR POSITION WALL SWITCH.
4. INTERLOCKING WIRING & CONDUIT PER DIVISION 16 SPECIFICATIONS.
5. SEE EXHAUST FAN SCHEDULE FOR INTERLOCKING.
6. PROVIDE CONTROL PANEL.

REFRIGERANT PIPING NOTES:

- A. REFRIGERANT LINE SIZES AND CONNECTIONS PER MFR'S RECOMMENDATIONS. MECHANICAL CONTRACTOR TO FIELD ROUTE REFRIGERANT LINES PER MFR'S RECOMMENDATIONS. (TYPICAL FOR ALL UNITS)
- B. ALL REFRIGERANT PIPING SHALL BE INSULATED. ALL EXPOSED PIPING SHALL BE PROVIDED WITH AN ALUMINUM JACKET.
- C. REFRIGERANT LINES EXCEEDING MANUFACTURER'S MAXIMUM LENGTH AND VERTICAL DIFFERENTIAL SHALL COMPLY WITH MANUFACTURER'S LONG-LINE APPLICATION GUIDELINES.

WSM ARCHITECTS
 4330 NORTH CAMPBELL AVE
 SUITE NO 268
 TUCSON, ARIZONA 85718
 520.408.1044 TEL
 520.408.1170 FAX
 WSMARCH.COM WEB

thompson design architects
 873 west 10th street
 tucson, arizona 85704
 520.233.1034
 520.233.1034
 520.233.1034

PROJECT:

REVISION:		
△	C.O.Y. COMMENTS	2/3/17
△	ADDENDUM #2	3/29/17
△		
△		
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PROJECT NO.:

DRAWN: LAS

CHECKED:

DATE:

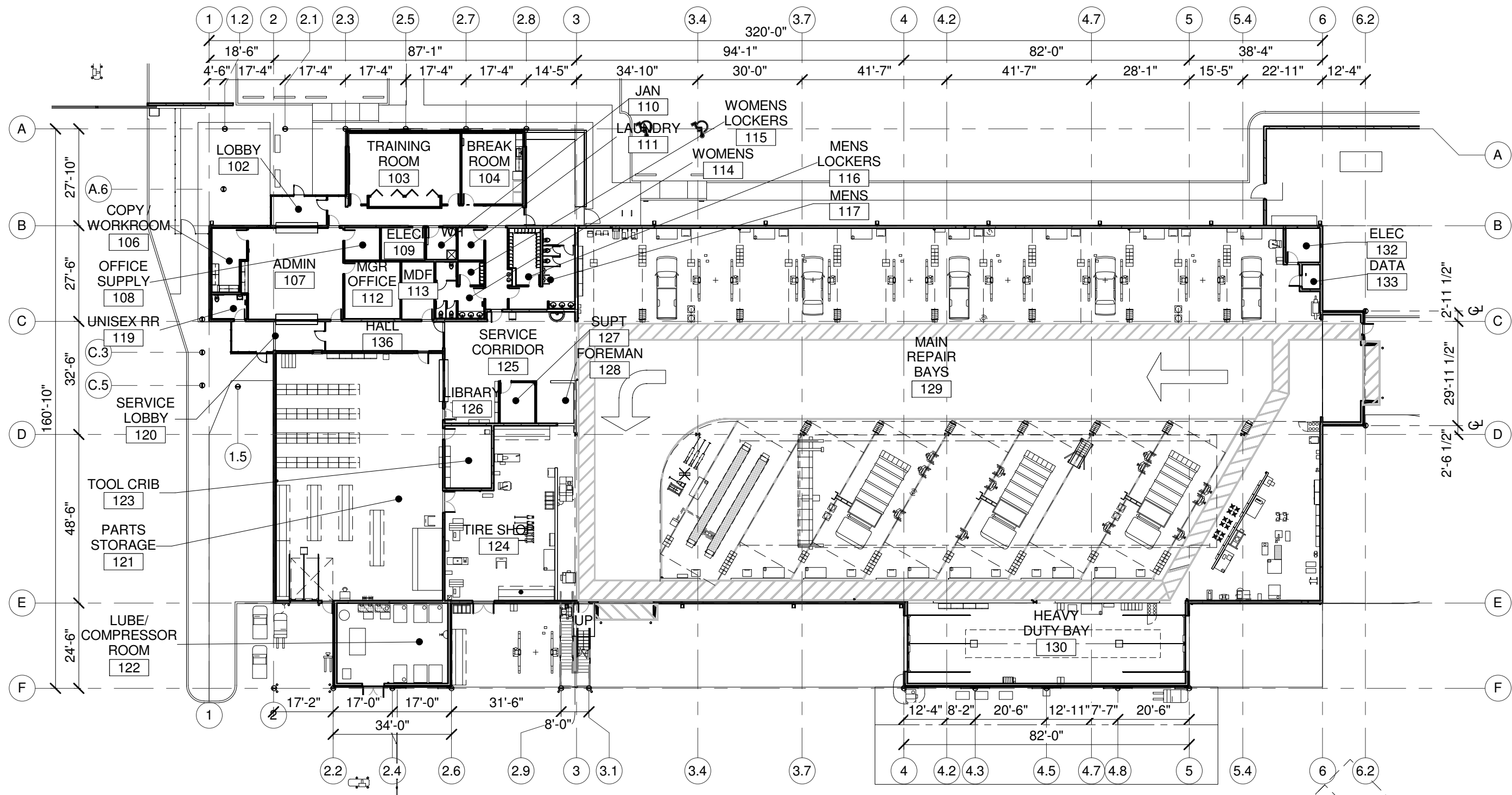
PILOT DATE:

SHEET TITLE:
MECHANICAL SCHEDULES

SHEET NO.:

M3.0





BUILDING MAX HEIGHT 30'-6"

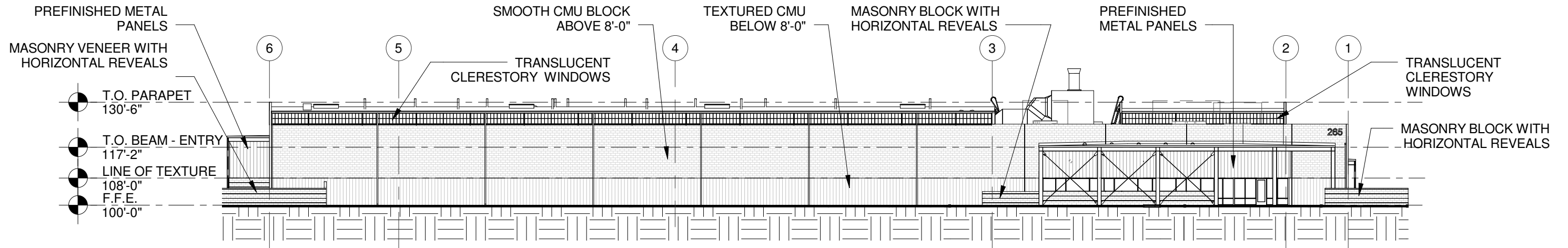
GROSS FLOOR AREA 38,236 SF
OVERHANG 2,397 SF

TOTAL AREA: 40,633 SF

MASONRY OPTION

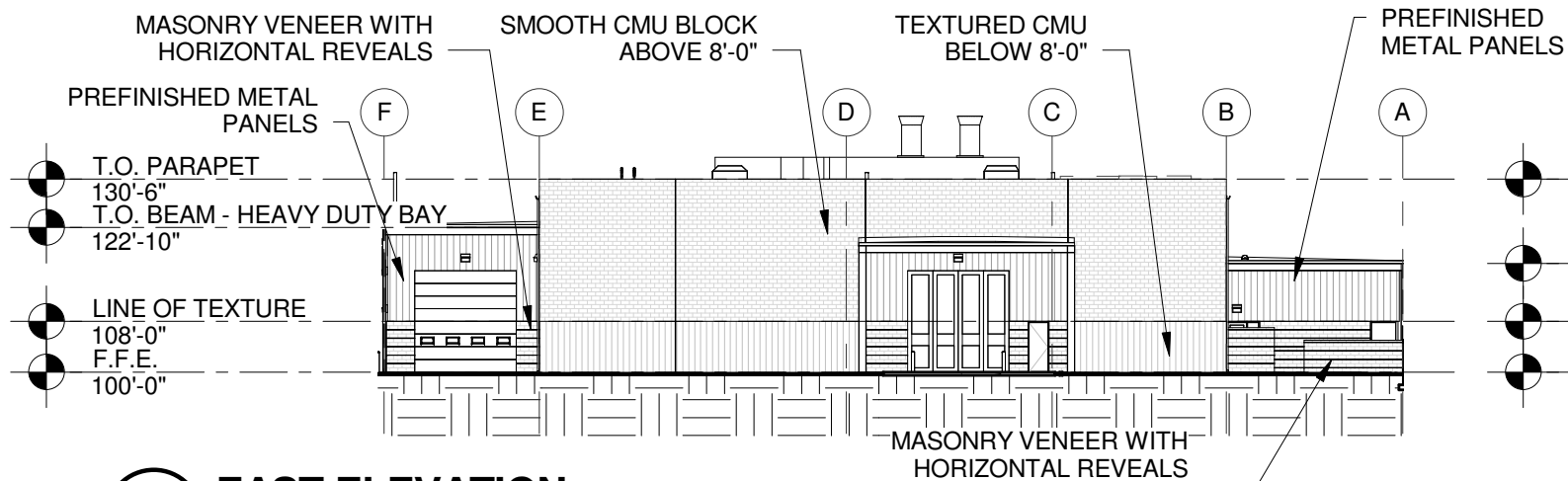


ARCHITECTS



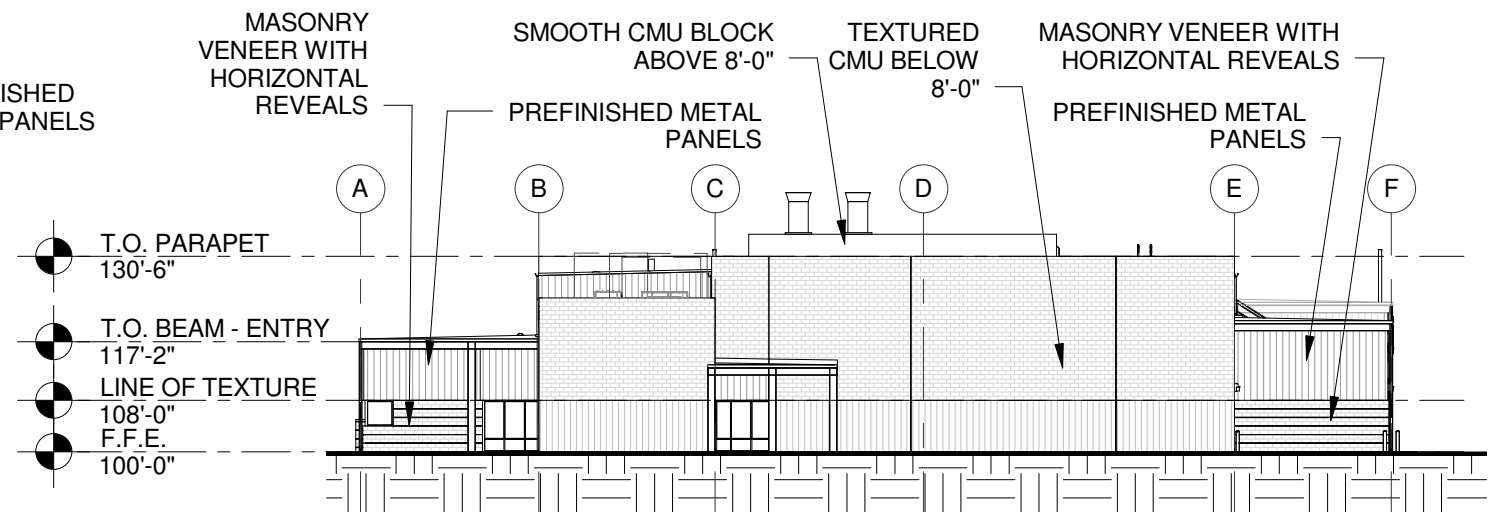
1 NORTH ELEVATION

1" = 30'-0"



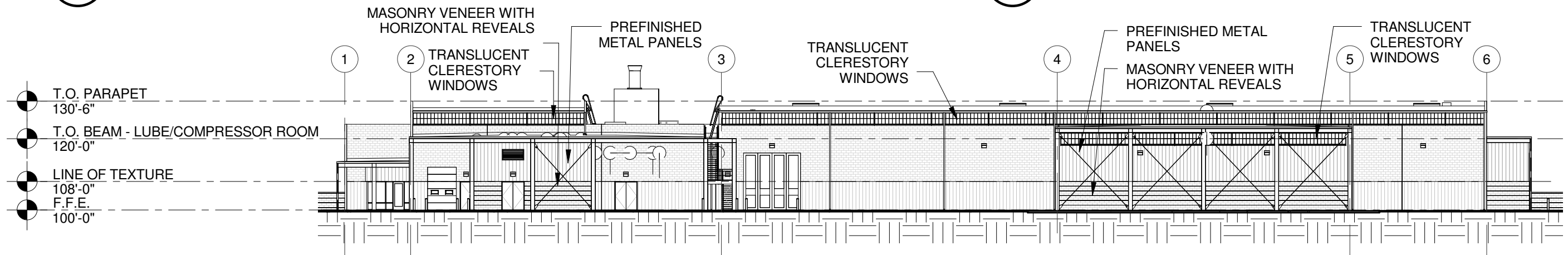
2 EAST ELEVATION

1" = 30'-0"



3 WEST ELEVATION

1" = 30'-0"



4 SOUTH ELEVATION

1" = 30'-0"

MASONRY OPTION